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# 1 Study of continuous cake pre-baking in a rectangular channel using ohmic 2 heating

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## 11 12 ABSTRACT

13 An original device dedicated to additive manufacturing was developed to pre-bake cake batter. Its  
14 originality lies in the fact that pre-baking is ensured by ohmic heating implemented in a rectangular  
15 channel equipped with two parallel electrodes. Experiments and numerical studies were carried out  
16 and benchmarked. The rheological properties of the cake batter (non-Newtonian power-law fluid) and  
17 the influence of voltage and temperature on electrical conductivity were accommodated in the  
18 numerical model. Due to the low velocities of the very viscous products near the solid-liquid interface,  
19 it was found that the heterogeneity of the temperature at the nozzle outlet under continuous ohmic  
20 heating could lead to nozzle clogging when high temperatures are reached. Hot spots were identified  
21 in different areas of the channel, such as corners where the electric field is high and velocity is close to  
22 zero. A parametric study was performed on the impact of the thermophysical properties of the batter,  
23 showing that specific heat has a much greater impact than thermal conductivity on the accuracy of the  
24 temperatures computed. Analysis of the process parameters showed that a stronger electric field leads  
25 to a higher temperature gradient in the nozzle section. The temperature gradient decreases with  
26 electrode distance ( $d_{elect}$ ) and nozzle width ( $l$ ). This model could be used to optimize the ohmic heating  
27 nozzle configuration, with the objective of obtaining a continuous flow of pre-baked cake batter, while  
28 preventing clogging. Such a system could be used as 3D printing head.

### 29 *Industrial relevance:*

30 Conventional 3D printing of bakery products is based on the deposition of a batter, followed by a  
31 baking. This study proposes an innovative approach based on a printing nozzle equipped with ohmic  
32 heating, with the objective of achieving a uniform temperature distribution and of obtaining high mass  
33 flow rate. Despite a significant temperature gradient in the baked batter at the exit of the nozzle, the  
34 obtained results showed that adjusting the nozzle geometry allows a reduction of the gradient. Further  
35 investigations are thus needed to reduce the temperature gradient and to accommodate expansion of  
36 the batter in the case of formulation using baking powder. The transfer of such concept for industry  
37 application may be doable in a close future.  
38

39 **Keywords:** Continuous Ohmic Heating, Baking, Cake Batter, Modeling, Electrical Conductivity,  
40 Rheology

### 41 42 Nomenclature

A	Electrode cell area for electrical conductivity measurement (m <sup>2</sup> )
C <sub>p</sub>	Heat capacity (J·kg <sup>-1</sup> ·K <sup>-1</sup> )
d <sub>elect</sub>	Electrode nozzle distance (mm)
D <sub>h</sub>	(4×S <sub>2</sub> )/P, Hydraulic diameter (m)
I	Current (A)

k	Thermal conductivity ( $\text{W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$ )
K	Constant of Eq. (14) ( $^{\circ}\text{C}^{-1}$ )
L	Nozzle length (mm)
l	Nozzle width (mm)
$L_h$	Hydrodynamic entry Length (m)
$L_{\text{elect}}$	Electrode cell distance for electrical conductivity measurement (m)
$m_d$	Mass of batter sample (kg)
$m_i$	Mass of batter when weighing immersed in oil (kg)
P	Nozzle perimeter (m)
m	Consistency index ( $\text{Pa}\cdot\text{s}^n$ )
$\dot{m}$	$S_2\times V_2$ , Volume flow rate ( $\text{kg}\cdot\text{s}^{-1}$ )
n	Flow index (-)
$P_r$	$(\mu C_p)/k$ , Prandtl number
$\dot{Q}$	Ohmic heating source ( $\text{W}\cdot\text{m}^{-3}$ )
S	Section ( $\text{m}^2$ )
$S_2$	nozzle surface in the normal direction of the flow ( $\text{m}^2$ )
$T_{\text{ref}}$	Reference temperature ( $^{\circ}\text{C}$ )
$T_{\text{m-N}}$	Nozzle mean temperature ( $^{\circ}\text{C}$ )
T	Temperature ( $^{\circ}\text{C}$ )
$V_1$	Piston velocity ( $\text{mm}\cdot\text{s}^{-1}$ )
$V_2$	Batter velocity inside the nozzle ( $\text{mm}\cdot\text{s}^{-1}$ )
V	Voltage (V)
Greek letters	
$\sigma$	Electrical conductivity ( $\text{S}\cdot\text{m}^{-1}$ )
$\sigma_{\text{ref}}$	Reference electrical conductivity at $T_{\text{ref}}$ ( $\text{S}\cdot\text{m}^{-1}$ )
$\rho_{\text{app}}$	Batter apparent density ( $\text{kg}\cdot\text{m}^{-3}$ )
$\rho_{\text{oil}}$	Oil density ( $\text{kg}\cdot\text{m}^{-3}$ )
$\mu$	Viscosity ( $\text{Pa}\cdot\text{s}$ )
$\Delta p$	Differential pressure (Pa)
Subscripts	
app	Apparent
d	Batter sample
elect	electrodes
h	Hydraulic, hydrodynamic
i	Immersed in oil
m-N	Nozzle mean value
oil	oil
ref	reference
1	Point to refer to piston region

43  
44  
45

## 1. Introduction

46 3D printing (3DP), also known as additive manufacturing (AM), has undergone rapid growth in the  
47 last few years, due to its large scope of application (Ali et al., 2016; Liu et al., 2018). It was first  
48 introduced by researchers from Cornell University, using hot-melt extrusion to create personalized  
49 chocolate products in complex 3D structures, using a Fab@home printer (Sun et al., 2015). 3D  
50 printing applied to food is based on different technologies, as recently reviewed by Le-Bail et al.  
51 (2020), and the most important applications are fused deposition modeling (FDM) and soft-material  
52 extrusion (Severini & Derossi, 2016). A variety of materials has been used to demonstrate food  
53 printing. Most of the existing publications linked to the 3D printing of baked products are based on the  
54 deposition of a batter baked after deposition. The earliest application of 3D printing focused on the  
55 printing of a “cake mix”, by extruding a batter mixture that consisted of starch, sugar, corn syrup,  
56 yeast, and a cake frosting (Lanaro et al., 2017; Yang et al., 2001). However, the biggest challenges are  
57 ingredient mixing, the associated rheology, structure accuracy and shape-stability, compatibility with  
58 traditional food processing technologies (e.g. baking and drying), and printing speed. Usual  
59 approaches to solidify the structure are based on cold extrusion, followed by a post-processing step.  
60 Lipton et al. (2010) demonstrated that traditional cookie recipes were compatible with 3D printing.  
61 However, extruded products do not retain their shape and structure after post-processing (e.g. baking),  
62 due to the presence of high amounts of fat. Two methods can be used to solve the shape stability  
63 problem: additives and recipe control (Lille et al., 2018; Lipton et al., 2015, 2010), but solutions  
64 consisting of pre-baking the product have not been explored. The challenge consists in solidifying the  
65 product while ensuring its flow. **Using a conventional heating system (based on hot walls) would yield  
66 a batter to crumb transition starting at the wall with a progressive transfer towards the center of the  
67 flow. This may cause several problems, in particular because the baked batter (crumb) would be much  
68 harder than the unbaked batter at center of the flow.**

69 This paper focuses on new process solutions like solidifying while printing to prevent shape  
70 destruction by implementing ohmic heating in the nozzle head. Ohmic heating (OH) is a thermal  
71 process consisting of the internal generation of heat by the passage of an electric alternating current  
72 (AC) through a medium with electrical resistance, such as food. In contrast to conventional heating  
73 (CH), where the heat of a hot surface is conducted from the outside of a food to its inside, OH is  
74 considered to induce heat within the entire mass, i.e. uniformly through the food (Leizeron &  
75 Shimoni, 2005). **OH should allow for higher mass flow rate of the printing nozzle, accurate control of  
76 the temperature and improvement of the energy efficiency.** This concept is not new and was used in  
77 the early 20<sup>th</sup> century for the electric pasteurization or sterilization of milk and other pumpable foods  
78 (viscous or liquid foods), such as fruit and vegetable products (juices, purees, pulps, etc.), and aseptic  
79 packaging (De Alwis & Fryer, 1990b; Sarkis et al., 2013; Yildiz & Guven, 2014). OH was also  
80 applied to solid foods and significant differences between OH and CH were highlighted. Textural  
81 changes can occur following the microscopic and macroscopic changes in food products (Gavahian et  
82 al., 2019). The kinetic of textural softening and the product texture could be affected by volumetric  
83 heating at high rate. Kamali & Farahnaky, (2015) have also shown that ohmically processed  
84 vegetables were characterized by a high textural cohesiveness. OH provides food industries with  
85 several benefits such as saving in process time and energy. One of the main advantages is the rapid  
86 and relatively uniform heating achieved (Marra et al., 2009). Contrary to classical heaters, whose  
87 heating is not homogeneous, due to the nature of heat migration, OH can achieve homogenous  
88 warming impossible to accomplish with classical heaters (Shynkaryk & Sastry, 2012). One important  
89 key for successful OH is first to identify possible hot and cold spots, in order to eliminate them and  
90 ensure the uniformity of heating or to achieve the desired temperature profile.

91 Several studies on ohmic cooking have been performed, but few of them have focused on bakery  
92 products. The first application in baking was carried out on bread by Baker, (1939), whose aim was to  
93 obtain a baked dough without temperature gradients. Gally et al., (2016) showed the interest of using  
94 OH to bake crustless bread in batch conditions. Studies on several applications of OH to the batch  
95 baking of cake have been published such as (e.g. Deleu et al., 2019; Luyts et al., 2013; Masure et al.,

96 2019). However, to our knowledge, the baking of bakery products by OH in continuous conditions has  
97 never been studied. In such conditions, with a strong change in the rheological properties of the batter  
98 during the batter-crumbs transition, OH appears as much more adapted than CH using a heated channel.

99 Mathematical modeling is a valuable tool for the development, understanding, and validation of  
100 these emerging thermal technologies (Tijskens et al., 2001). It allows evaluation of the influence of  
101 key variables such as electrical field strength and sample conductivity. Initial models of ohmic  
102 processes, mainly two-dimensional systems, were developed for continuous flow systems, using  
103 liquid-solid mixtures (Marra et al., 2009). Numerical modeling is essential to understand the OH  
104 process, due to the difficulty of measuring real internal temperatures during continuous flow, because  
105 of the presence of a strong electric field (Sastry & Palaniappan, 1992). Mathematical models were  
106 developed and validated experimentally to study the heating patterns of solid-liquid mixtures in static  
107 ohmic heaters (De Alwis & Fryer, 1990a; Salengke & Sastry, 2007; Sastry & Palaniappan, 1992; Shim  
108 et al., 2010; Zhang & Fryer, 1993). On the other hand, numerical models for multiphase foods in  
109 continuous ohmic heaters have been developed, due to the complexity of the parameters (Chen et al.,  
110 2010; Choi et al., 2014; Sastry, 1992). Some works have concerned the continuous OH treatment of  
111 highly viscous fluids (Shynkaryk & Sastry, 2012). The numerical results have shown the importance  
112 of the OH chamber geometry, as it influences both current and fluid flow patterns and thus the  
113 uniformity of heating.

114  
115 The present study focuses on a new approach, consisting of the partial baking of a cake batter,  
116 using ohmic heating embedded inside the printing nozzle head. This study is comprised of three main  
117 parts. First, an original printing nozzle device, based on a rectangular canal, was developed to obtain a  
118 cake batter pre-baked by OH. Second, the thermophysical and electrical properties of the cake batter  
119 were determined, and then a numerical model was developed to model OH in continuous flow, with  
120 the objective of understanding the links between the thermophysical properties, the channel geometry,  
121 and the temperature profile in the printing nozzle. After validating the numerical model, additional  
122 simulations were carried out to establish recommendations to ensure convenient cake pre-baking by  
123 continuous OH.

## 124 2. Materiel and Methods

### 125 2.1. Materials

#### 126 2.1.1. Rectangular nozzle description

127 A rectangular nozzle was chosen because of its ease of development and versatility. 3D geometry  
128 was plotted on Autodesk® Inventor® 2015; the drawing of the nozzle is presented in Fig. 1. It consists  
129 of: 1) two parallel stainless steel electrodes (red), enclosed in 2) two rectangular polycarbonate plates  
130 (gray), separated by 3) two polycarbonate beams (green) 0.105 m long, with variable width and  
131 thickness to allow calibrating the nozzle dimension, and finally, 4) a channel support also made of  
132 polycarbonate (A) and (B). Fig. 1 (C) shows the nozzle dimensions, where the nozzle length,  $L$ , is  
133 equal to 105 mm. The total width of the device,  $b$ , is equal to 80 mm; the thicknesses of the electrodes  
134 and the polycarbonate plates are equal to 1 mm and 10 mm, respectively. The nozzle section depends  
135 on the width,  $l$ , and on the distance between the electrodes,  $d_{elect}$ , set at 4 cm and 1 cm, respectively,  
136 for this paper.

#### 137 2.1.2. System description

138 The complete experimental set-up is presented in Fig. 2 (A). A drive motor (H T23-400D, ORIGA  
139 System plus, USA) was used to (1) impose the displacement of the piston at a constant velocity, (2)  
140 ensure a flow of batter in the cylindrical container (3), and then in the rectangular nozzle (4). The  
141 nozzle was equipped with electrodes to heat and pre-bake the cake batter.

142 The voltage applied between the electrodes (ohmic heating) by the voltmeter (a) (Fluke 45 dual  
143 display multimeter, USA) was controlled by an autotransformer (d) (DEREIX S.A, Paris, rotortransfo  
144 120 NC, reference R.212). All the data collected from the experiments, especially the temperature  
145 variations measured between the inlet and the outlet, were logged by a data logger (c) (AOIP, France)  
146 and stored in the PC (f) (logidat program, Samsung computer). The current flowing through the cake  
147 batter was recorded with an ammeter (b) (Multimeter MN 5128, AOIP Mesures, France).

148 During each experiment, the velocity applied by the piston (2) and the voltage between the  
 149 electrodes were constant, and the temperature of the batter through the nozzle increased due to ohmic  
 150 heating.

151 *2.1.3. Experimental procedure and outlet temperature*

152 All the experiments were conducted in such a way that the batter passing through the nozzle did not  
 153 solidify. The nozzle dimensions were mentioned in section 2.1.1, and the process parameters were set  
 154 to 65 V for the voltage and  $V_1 = 0.1$  mm/s for the piston speed. Four thermocouples ( $T_1$  to  $T_4$ ) were  
 155 installed at the outlet of the nozzle (Fig. 2B), and one thermocouple  $T_5$  at the inlet, to measure the  
 156 variation of temperature along the nozzle. Four experiments were carried out with the same process  
 157 parameters, and the temperature of the cake batter at the inlet was between 16 °C and 22 °C.

158 *2.1.4. Cake batter sample preparation*

159 Cake batter was prepared based on the recipe and cake-making method of Hesso (Hesso et al.,  
 160 2014), except that no baking powder was used. The recipe used for the reference cake batter is given  
 161 in Table 1. The ingredients of each formula were mixed in a KitchenAid (KSM90, KitchenAid, St.  
 162 Joseph, MO, USA) in two stages to obtain the final cake batter. In the first stage, sugar, fat, and liquid  
 163 egg were mixed for 2 min at average speed 6, then the wheat flour and salt were added and mixed in  
 164 the second stage for 3 min at high speed 8.

165  
 166 **Table 1**  
 167 Cake batter recipe.

Ingredient	Origin	Percentage wb(%)	Water content (% db) [ISO 712 : 2001(1998)]
<b>Wheat Flour T55</b>	Petits moulins de France		
<b>Sugar (S)</b>	Saint Louis, France	25	
<b>Fat (FA)</b>		20	
<b>Liquid Egg (E)</b>	Ovoteam, France	25	
<b>Salt (SA)</b>	Cerebros, Esco France s.a.s, France		

168 *2.2. Batter characterization*

169 *2.2.1. Ohmic heating system and measurement of electrical conductivity*

170 A literature review showed the importance of controlling electrical conductivity during OH of a  
 171 food product. In order to correctly design and control the batter cooking process, it was essential to  
 172 determine the batter's electrical conductivity under different experimental conditions. All experiments  
 173 were carried out in triplicate.

174 The ohmic cell, used to calculate the electrical conductivity of liquid products, was adapted from Gally  
 175 et al. (2016), with small changes in cell material and dimensions. The short cell was replaced by a new  
 176 polyacetal with an internal diameter of 29 mm and an external diameter of 42 mm.

177 The impact of electric voltage on the electrical conductivity of the cake batter was monitored with  
 178 AC voltages (50 Hz) of 100 V, 150 V, and 200 V (22 V/cm, 33 V/cm, and 48 V/cm). The time–  
 179 temperature data were plotted to obtain the ohmic heating curves for the batter. Electrical conductivity  
 180 was plotted against the corresponding temperature to obtain the electrical conductivity curves.  
 181 Electrical conductivity was measured using the following equation:

$$\sigma = \frac{(L_{elect} \times I)}{(A \times V)} \quad (1)$$

183

184  $L_{elect}/A$  denotes the cell constant and is equal to  $67.72 \text{ m}^{-1}$ . It is calculated from a calibration, using the  
185 method of Olivera et al. (2013). Solutions of potassium chloride and sodium chloride were used as  
186 references, for which their electrical conductivities are known for their temperatures and  
187 concentrations.

### 188 2.2.2. Measurement of thermal conductivity and heat capacity

189 The thermal conductivity of the cake batter was measured based on the method proposed by Sweat  
190 & Parmelee (1978) and also used by Jury et al. (2007), using a line-heat source probe. The probe  
191 consists of a hypodermic needle with an external diameter of 0.7 mm and a length of 3.5 cm,  
192 containing a heating wire and a K-type thermocouple (OMEGA, Stamford, CT, USA). The thermal  
193 conductivity measurements were performed at different temperatures, by placing the samples in a  
194 climatic chamber (Froilabo, Meyzieu, FRANCE).

195 The heat capacity was measured by micro-calorimetry, with a micro DSC VII (SETARAM,  
196 Caluire, FRANCE), calibrated against the specific heat of saphir. Around 100 mg of batter was  
197 installed in a stainless steel pan, which was hermetically closed. A heating rate of  $1 \text{ }^\circ\text{C}/\text{min}$  was used  
198 from  $10$  to  $120 \text{ }^\circ\text{C}$ ; the integration of the thermogram provided an enthalpy function whose derivative  
199 yielded a specific heat function (provided as a function of temperature). All the thermal conductivity  
200 and heat capacity experiments were carried out in triplicate.

### 201 2.2.3. Density

202 A densimetric method was used to calculate the apparent density of the cake batter (Baker & Mize,  
203 1946). A 500 ml beaker was filled with rapeseed oil (density:  $935.5 \text{ kg}/\text{m}^3$ ) and placed on a scale. A  
204 holder was maintained in the oil, and the tare was set. The batter was weighed on the scale and then  
205 immersed in the oil. Its apparent density was calculated using the following equation:

$$206 \rho_{app} = m_d \times \rho_{oil}/m_i \quad (2)$$

207  
208 The density of the oil was determined by the weight/volume method, using a  $45.28 \times 10^{-6} \text{ m}^3$  glass  
209 cylinder. The cup was filled with oil, leveled on the surface using a spatula, and weighed on an  
210 analytical balance. This technique yielded a true batter density, accommodating the possible presence  
211 of gas entrapped in the batter during mixing.

### 212 2.2.4. Rheological properties

213 The hydro-thermal transformations undergone by starch and gluten are the origin of the evolution  
214 of the rheological properties of a dough during cooking (Bloksma, 1980). Many models have been  
215 developed, leading to a large number of phenomenological models for dough viscosity (Zhou, 2004).

216 The apparent viscosity of batter was determined using a rotational viscometer, 'VT550  
217 RHEOWIN' (HAAKE, France). The fluid was placed between two coaxial cylinders with a well-  
218 defined imposed shear rate, and the resulting shear stress was measured. Rheological measurements  
219 were performed between  $25$  and  $95 \text{ }^\circ\text{C}$ , with a precision close to  $\pm 1\%$  for the apparent viscosity, using  
220 a Ministat 230/Huber with a step of  $10 \text{ }^\circ\text{C}$ . The variation of the consistency index,  $m$ , and the flow  
221 index,  $n$ , of the batter was determined for a shear rate between  $0.9$  and  $150 \text{ s}^{-1}$  in 2 min. These  
222 experiments were performed in triplicate.

## 223 2.3. Numerical models

### 224 2.3.1. 2D and 3D geometries

225 2D and 3D numerical models were developed on COMSOL Multiphysics 5.4, to study the  
226 temperature profile of the cake batter in continuous flow at the nozzle outlet.

227 The 3D model is represented with symmetry along the  $xz$ -plane, to decrease simulation time. The  
228 2D geometry shows the nozzle in the  $xz$ -plane.

229 A mesh independency study for a laminar non-Newtonien fluid flow was performed to determine  
230 its effects on the CFD (Computational fluid dynamics) simulation results. It was clear that the  
231 simulation time was highly dependent on the number of mesh nodes, and a compromise had to be  
232 found between the grid-independent solution and the calculation time.

233 The numbers of domain elements chosen for the 2D and 3D models were equal to 3341 and 36238,  
234 for simulation times of 302 s and 1407 s, respectively.

### 235 2.3.2. Governing equations

236 Due to the existence of a strong electric field inside the nozzle, the temperature profile was very  
237 difficult to measure experimentally. The fluid temperature was measured at three points at the outlet of  
238 the nozzle. To carry out a stricter validation of numerical models, we made the choice to study the  
239 non-stationary flow, in order to obtain the evolution of the temperature at the nozzle outlet.

240 Navier–Stokes equations were used to describe the incompressible fluid flow, in which flow  
241 behavior is governed by:

$$\rho \partial u / \partial t + \rho (u \cdot \nabla) u = \nabla \cdot [-pI + \mu(\nabla u + (\nabla u)^T)] + F \quad (3)$$

242 where  $u$  is the velocity field. From left to right, the different terms in the equation above correspond to  
243 the inertial forces, the pressure forces, the viscous forces, and the gravitational force applied to the  
244 fluid ( $F = -\rho g$ ).

245 Besides the Navier-Stokes equation, the continuity equation was also solved:

$$\partial \rho / \partial t + \rho \nabla \cdot (u) = 0 \quad (4)$$

247 where  $\partial \rho / \partial t = 0$ , because it is considered to be an incompressible fluid.

249 In the case of the continuous flow ohmic heating process, the product is subjected to temperature  
250 and shear-rate gradients. A power-law model is used to fit the batter viscosity:

$$\mu = m \dot{\gamma}^{(n-1)} \quad (5)$$

253 where  $\mu$  is the viscosity,  $\dot{\gamma}$  the shear rate,  $m$  the consistency index which is temperature dependent, and  
254  $n$  the flow behavior index.

255 Heat transfer within the liquid sample was solved and based on the general heat equation with a source  
256 term:

$$\rho C_p \partial T / \partial t + \rho C_p u \cdot \nabla T = \nabla \cdot (k \nabla T) + \dot{Q} \quad (6)$$

259 where  $\rho$  is the density which is constant,  $C_p$  is the heat capacity at constant pressure, and  $k$  is the  
260 thermal conductivity.  $C_p$  and  $k$  are temperature dependent, and is the Ohmic heating power source.  
261 The heat, in Eq. (7) is generated by Joule effect, due to the dissipation of electrical energy into heat:

$$\dot{Q} = \sigma \cdot |\nabla V|^2 \quad (7)$$

264 where  $|\nabla V|$  represents the modulus of the gradient of electrical potential, and  $\sigma$  is the temperature-  
265 dependent electrical conductivity of the batter.

266 The electrical potential distribution within the batter was computed using the following Laplace  
267 equation:

$$\nabla \cdot \sigma \nabla V = 0 \quad (8)$$

### 269 2.3.3. Boundary and initial conditions

270 A vertical natural convection condition was considered to exist at all the faces of the nozzle. The  
271 heat losses, due to convection phenomena between the nozzle and the ambience around the sample,  
272 take into consideration the external heat-transfer coefficient ( $h_{ext}$ ), using COMSOL equations with  $q$   
273 being the convection heat flux ( $W \cdot m^{-2}$ ):



$$q = h_{ext} (T_{ambient} - T) \quad (9)$$

274

275 The initial batter temperature is equal to  $T_i$  at  $t = 0$  (0 - time injection), and the initial nozzle  
 276 temperature is equal to  $T_{ambient} = 20$  °C.

277 For the inlet flow conditions, it was shown that a uniform velocity profile at the inlet affects the  
 278 temperature profile as well as the pressure drop along the nozzle. An analysis was performed to choose  
 279 the right velocity profile at the nozzle inlet and to calculate the hydrodynamic entry length ( $L_h$ ),  
 280 Reynolds number ( $N_{GRe}$ ), and pressure drop ( $\Delta p$ ). The calculation was performed by considering the  
 281 mean values of the different parameters at the mean nozzle temperature between the inlet and the  
 282 outlet. The power-law Reynolds number used is:

$$N_{GRe} = 2^{(3-n)} \frac{\rho_{app} V_2^{2-n} D_h^n}{m \left(\frac{3n+1}{n}\right)^n} \quad (10)$$

283

284 The hydrodynamic entry length of this region was calculated from:

285

$$L_h = 0,05 R_e D_h \quad (11)$$

286

The differential pressure of power-law non-Newtonian laminar flow is given by:

$$\Delta p = \frac{2^{n+2} \left(\frac{3n+1}{n}\right)^n L m V_2^n}{(D_h)^{n+1}} \quad (12)$$

287

Table 2 shows the analytical results used as boundary conditions for the numerical study.

288

289

290

**Table 2**

Rectangular nozzle – 65 V –  $V_1 = 0.1$ mm/s - analytical study.

Experience		$T_{m-N}$ *(°C)	$N_{GRe}$	$L_h$ (m)	$\Delta p$ (Pa)
$V_2$ (mm/s)	V	Expe	Eq (10)	Eq (11)	Eq (12)
4	65	33.33	0.0023	$1.8410^{-6}$	1415.1

291

\* mean nozzle temperature between the nozzle inlet and outlet.

292

293 The  $L_h$  ( $1.84 \cdot 10^{-6}$  m) obtained is much shorter than the nozzle length (0.105 m). As a consequence,  
 294 the velocity was fully developed at the nozzle entrance section, as this region was much longer in the  
 295 laminar flow condition than in turbulent flow (Cengel, 2008). The batter was characterized by a high  
 296 Prandtl number ( $P_r = 421973$ ), so that the flow is considered as thermally developing but  
 hydrodynamically developed.

297

298 The wall conditions (solid – liquid interface). For the solid-liquid interface, no slip condition was  
 299 considered to exist inside the nozzle, and the flow velocity in the immediate vicinity of the surface was  
 assumed to be zero.

$$u = 0 \text{ at } \forall t \geq 0 \quad (13)$$

300

301 At the nozzle outlet section, the atmospheric pressure condition was set at  $p = 0$  for any time above  
 302 the initial time ( $\forall t \geq 0$ ).

303

### 3. Results and Discussion

304 3.1. Effect of temperature and voltage gradient on electrical conductivity

305 Eq. (1) was used to determine the electrical conductivity of the cake batter and its evolution with  
 306 temperature, as shown in Fig. 3 (A), at a voltage gradient of 50 V/cm.

307 Three stages are distinguishable. Zone 1 shows a polynomial curve of the second degree ( $R^2 >$   
 308 0.999), before starch gelatinization, from 26°C to 80°C; at this stage, the variation of electrical  
 309 conductivity can be calculated, using a reference temperature, and expressed by Eq. (14). These results  
 310 can be compared with those of Gally et al. (2016), who observed a linear variation of electrical  
 311 conductivity, before starch gelatinization, for bread dough. Zone 2 begins at just below 80 °C, which  
 312 represents the temperature at the beginning of starch gelatinization. From there, the rate of increase of  
 313 electrical conductivity decreases strongly, until the temperature reaches 98°C, and at some stage, it  
 314 becomes almost constant. Zone 3 shows a linear variation of electrical conductivity, with a slope  
 315 different from that of zone 1. Similarly, Gally et al. (2016) obtained a linear pattern after  
 316 gelatinization, with a different slope ( $d\sigma/dT$ ), and attributed it to a change in the product.

317 The changes in the electrical conductivity of batter with temperature, during ohmic heating at three  
 318 different voltage gradients, are shown in Fig. 3 (B). For a voltage gradient of 48 V/cm, the onset  
 319 temperature of starch gelatinization was 80 °C, that of 33 V/cm was 90°C, and that of 22 V/cm was  
 320 100°C. These experiments showed that electrical conductivity increases with voltage gradient. Similar  
 321 observations were reported for tomato juice in the range of 50–70 V/cm (Srivastav & Roy, 2014), and  
 322 for apricot and peach juices in the range of 20–70 V/cm (Icier & Ilicali, 2005). As the voltage gradient  
 323 increased, the heating time of the cake batter, required to reach the prescribed temperature, decreased.  
 324 It was found that the ohmic heating time increased by up to 3.6 times, when the voltage gradient was  
 325 decreased from 48 to 22 V/cm (Fig. 3C).

326 Electrical conductivity was modeled in zone 1 (before starch gelatinization) and can be written as  
 327 follows:

$$\sigma = \sigma_{ref} (1 + K'(T - T_{ref})^2 + K(T - T_{ref})) \quad (14)$$

328 where  $K$  and  $K'$  are constants, and  $T_{ref}$  and  $\sigma_{ref}$  are the reference temperature and electrical  
 329 conductivity, respectively. Their values, at the three different electric voltages, are presented in Table  
 330 3. Fig. 3 (B) was used to model the variation of electrical conductivity with temperature at different  
 331 voltages.

332  
 333 **Table 3**  
 334 Values of reference temperature, electrical conductivity, and constants  $K$  and  $K'$ .

Parameter	200V	150V	100V
$T_{ref}$ (°C)	26.3	26.3	26.3
$\sigma_{ref}$ (S/m)	0.048	0.051	0.055
$K$ (°C <sup>-1</sup> )	0.04	0.034	0.024
$K'$ (°C <sup>-2</sup> )	$3.5410^{-4}$	$2.110^{-4}$	$1.4710^{-4}$

335

336 3.2. Thermophysical properties of cake batter

337 3.2.1. Viscosity profile

338 Fig. 4 (A) and Fig. 4 (B) show the data, in logarithmic form, for the cake batter sample at a  
 339 reference temperature of 25°C, and at different temperatures (35°C, 45°C, 75°C, and 85°C),  
 340 respectively, where the shear rate varied from 0.9 to 150 s<sup>-1</sup>. It was found that the power law fits the  
 341 apparent viscosity behavior very well ( $r^2=0.99$ ). The results illustrate the power-law non-Newtonian  
 342 behavior of the cake batter. It is clear, in Fig. 4 (B), that the viscosity decreases, as expected (Bloksma,  
 343 1980), when the batter temperature increases at constant shear rate, before the gelatinization onset  
 344 temperature. At a shear rate of 0.5 s<sup>-1</sup>, the viscosities are equal to 1.4 and 0.8 Pa.s, at 25 °C and 75 °C,  
 345 respectively. After gelatinization at 85°C, for the same shear rate of 0.5 s<sup>-1</sup>, the viscosity increases  
 346 again and is equal to 0.95 Pa.s.

347 Flow index,  $n$ , is equal to  $0.6 \pm 0.05$  and is considered constant. The consistency index,  $m$ , is  
348 variable with temperature; it is equal to the viscosity, for a shear rate of  $1 \text{ s}^{-1}$ , as shown in Fig. 4 (C).  
349 Its evolution as a function of temperature is represented by Eq. (15).  
350

$$m = 9 \times 10^{-5} \times T^3 - 0,0063 \times T^2 - 0,6672 \times T + 58,203 \quad (15)$$

351  
352 It can be seen, in Fig. 4D, that the viscosity decreases with temperature at a fixed shear rate, until it  
353 reaches the gelatinization temperature. After that, it increases with temperature up to  $95^\circ\text{C}$  at a  
354 different shear rate. The viscosity decreases when the shear rate increases at constant temperature. At  
355  $25^\circ\text{C}$ , it is equal to 18 and 12 Pa. s, at shear rates of 10 and  $30 \text{ s}^{-1}$ , respectively.

### 356 3.2.2. Density, heat capacity, and thermal conductivity

357 The cake batter density ( $\rho$ ) was found to be  $970 \pm 8 \text{ kg. m}^{-3}$ . The cake batter thermal conductivity  
358 vs. temperature could be modeled using Eq. (16); the limit of validity of this equation is  $85^\circ\text{C}$ , which  
359 is the gelatinization onset temperature.

$$k(T) = 3,8 \times 10^{-5} \times T^2 - 1,27 \times 10^{-3} \times T + 0,21 \quad (16)$$

360  
361 The evolution and uncertainty of the cake batter's thermal conductivity and heat capacity with  
362 temperature are presented in Fig. 5.

363 Heat capacity and thermal conductivity are responsible for the considerable non-linearity of the  
364 numerical model, due to their temperature dependence.

### 365 3.3. Comparison between numerical simulation and experimental results

#### 366 3.3.1 Outlet nozzle temperatures – voltage 65 V – velocity 4 mm/s

367 The experimental and simulation results for the outlet nozzle temperature vs. time are presented in  
368 Fig. 6. 2D and 3D simulations showed good agreement with the experiments for temperature  $T_1$ . In 2D  
369 geometry, edge temperature  $T_2$  could not be determined, so a 3D simulation was necessary, which also  
370 showed a good correlation with the experiments.

371 The total injection time was 12 min at 65 V, with a batter velocity of 4 mm/s. When the steady state  
372 was reached after almost 7 min of injection, the results showed a temperature **difference** between  $T_1$   
373 and  $T_2$  equal to  $11^\circ\text{C}$ . The transition phase, from 0 to 7 min, showed a small difference between the  
374 simulation and the experimental results and was composed of two parts. The first was from 0 to 1 min,  
375 where the linear temperature evolutions of  $T_1$  and  $T_2$  with time occurred at the same rate of  $0.2^\circ\text{C/s}$ .  
376 The second was from 1 min to 7 min, where the temperature evolution was polynomial, and the rate of  
377 increase of  $T_2$  was faster than that of  $T_1$ . The temperature difference between  $T_1$  and  $T_2$ , along the x  
378 axis, was almost insignificant and equal to  $0.2^\circ\text{C}$ .

379 In previous studies, Marra et al., (2009) and Gally et al., (2016) showed the advantage of using  
380 ohmic heating in static conditions, with a homogeneous temperature distribution with higher  
381 temperature at the center, due to volumetric heating from inside to outside. In the present study  
382 involving continuous OH, the temperature gradient between the center and the side of the rectangular  
383 channel was reversed, with hot spots in the corners of the channel. To better understand the reason for  
384 this heterogeneity, it is necessary to understand what occurs inside the nozzle, with the help of the  
385 numerical model.

386 Several simulation results are presented in Fig. 7. Fig. 7 (A) shows the volume electric field. The  
387 points where the electric field is the strongest are those at the edges of electrodes. At the center of the  
388 electrodes, the electric field is equal to  $65 \text{ V/cm}$  (in blue). At the bottom of the electrode (horizontal  
389 line, parallel to the y axis in Fig. 7 (A) and Fig. 7 (B)), the electric field varies between 200 and 250  
390  $\text{V/cm}$ , and at the edge line (or vertical line, light blue in Fig. 7 (A)) parallel to the z axis, it is equal to  
391  $100 \text{ V/cm}$ .

392 The xy plane of the nozzle (Fig. 7B) also shows a strong electric field at the corner of the electrode  
393 (red zone). It can be seen that, inside the nozzle, the electric field passes through the batter in the  
394 perpendicular direction of the two electrodes. It can be seen that the electric field that crosses through

395 the nozzle body (polycarbonate) is negligible. The shear rate inside the nozzle depends on the velocity.  
396 Fig. 7 (C) shows that the lowest shear rate, which is almost equal to zero, is at the center of the nozzle,  
397 while the highest shear rate, which is parallel to the y axis and equal to  $4.89 \text{ s}^{-1}$ , is at the outlet edge of  
398 the nozzle.

399 The velocity profile tends to zero near the walls (solid-liquid interface) along the y and x axes, as  
400 shown in Fig. 7 (D). The residence time and the difficulty to remove heat by convection close to the  
401 walls cause temperature heterogeneity and higher temperature at the wall interface. As expected, the  
402 latter along the y-axis could cause fouling by clogging, due to starch gelatinization in the batter, in the  
403 case of higher voltages.

404 It is important to note the temperature distribution in the xy plane, as shown in Fig. 7 (E). It is  
405 obvious that the highest temperatures are at the corners between the electrodes and the polycarbonate  
406 beam. The temperature near the electrodes in the x direction is lower than those near the wall in the y  
407 direction, although the velocity is zero for both. This could be explained as being due to the strong  
408 electric field at these corners (Fig. 7B), along with very high viscosity and zero velocity. Hashemi &  
409 Roohi, (2019), have performed numerical modeling of process and bacterial inactivation kinetics with  
410 ohmic heating. They have found that the initiation of temperature rise near the regions with high  
411 gradient values of electric field was predictable with CFD. Jun & Sastry, (2005) have also found that  
412 the electric field strength near the edges of electrodes goes close to the maxima: up to  $4985 \text{ V/m}$  in  
413 their 2D numerical study on the ohmic heating of foods inside a flexible package. They conclude that  
414 the electrode configuration could be optimized to ensure uniformity of heating.

415 In summary, the heterogeneity of temperature is a consequence due not only to zero velocity at the  
416 wall interfaces, but also to the hot spots that occur at the electrode corners, because of the strong  
417 electric field and the very low velocity at these points.

418 In this section, the model developed was validated successfully, showing good agreement between  
419 the experimental and predicted temperatures. The main goal here was to predict the temperature  
420 profiles of the cake batter inside the nozzle. The results showed heterogeneous temperatures that could  
421 cause nozzle clogging at higher temperatures. The challenge now was to flatten and even reverse the  
422 temperature profile, by trying different process parameters and by studying the effect of batter  
423 properties on the outlet temperature profile. To do that, it seemed necessary to study the influence of  
424 certain parameters, and to determine their relative impact on the computed values.

### 425 3.3.2. Parametric study

426 A parametric study was carried out to monitor the impact of the batter properties and nozzle  
427 dimensions on the computed temperatures. In this section, all the results were obtained for stationary  
428 flows.

#### 429 A) Thermal conductivity

430 The effect of thermal conductivity was studied first. The results are shown in Fig. 8 (A). The  
431 experimental results showed that the thermal conductivity varied from  $0.2$  to  $0.4 \text{ W/(m. K)}$ , for a  
432 temperature range of  $24^\circ\text{C}$  to  $74^\circ\text{C}$ . A reference value was set to  $0.4 \text{ W/(m. K)}$ . When this value  
433 decreased by 50%,  $T_2$  increased by 3%, and  $T_1$  decreased by about 2.5%. When the thermal  
434 conductivity increased by 50%,  $T_2$  decreased by 2%, and  $T_1$  increased by 2%. It can be seen here that,  
435 at a higher thermal conductivity,  $T_1$  at the center and  $T_2$  at the near edge were reversed. It was  
436 observed that, when the thermal conductivity was increased by about 50%, the temperature gradient  
437 decreased by almost 5%. In summary, when the thermal conductivity increased, the temperature  
438 gradient between the center and the near edge decreased; this result was expected, since high thermal  
439 conductivity facilitates heat diffusion within the batter, resulting in better temperature equilibration  
440 during heat-up. In turn, a reduced temperature gradient was observed.

441 These results showed that the accuracy in determining the thermal conductivity of the batter had a  
442 minimal impact on the computed temperature, as a 50% increase in this property yielded only a 2% to  
443 3% uncertainty in the temperatures.

#### 444 B) Specific heat

445 The impact of specific heat was investigated in a similar way. It was observed that, when the  
446 specific heat increased, temperatures  $T_1$  and  $T_2$  tended to decrease, the lowest specific heat resulting, as  
447 might be expected, in the highest temperatures. When the heat capacity was decreased by 8%,

448 temperatures  $T_1$  and  $T_2$  increased by 4%, and when the heat capacity was increased by 19 %,  $T_1$  and  $T_2$   
449 decreased by 7%. A further result was that the specific heat had no effect on the temperature gradient,  
450 because  $T_1$  and  $T_2$  were affected at the same rate.

451 The temperature profile was not reversed, as had been shown in the previous study on the impact of  
452 thermal conductivity. The main finding was that greater accuracy in the determination of specific heat  
453 was needed, compared to that for thermal conductivity. For example, a deviation of 8% for specific  
454 heat led to a 4% deviation in the computed temperatures. However, the accuracy of specific heat was  
455 sufficient to conveniently predict the profile temperatures.

#### 456 C) Density

457 The reference density of the cake batter was 980 kg/m<sup>3</sup>. The density of the gas-free batter (obtained  
458 by computation) was 1200 kg/m<sup>3</sup>, thus indicating that the batter porosity was 24.3%. For a gas-free  
459 batter, the temperature observed at the center ( $T_1$ ) decreased from 37.7 °C to 34.6 °C, corresponding to  
460 a temperature difference of 8%. In contrast, when the batter density was decreased by 50% (from 980  
461 kg/m<sup>3</sup> to 500 kg/m<sup>3</sup>), it was observed that  $T_1$  increased by about 13%, based on the initial batter  
462 temperature. Here also,  $T_1$  and  $T_2$  were found to change at the same rate with density. Unlike thermal  
463 conductivity, but like thermal capacity, the batter density did not flatten the temperature profile.  
464 However, it seemed to be important to determine the batter density with sufficient accuracy, at least  
465 greater than that of the thermal conductivity, but less than that of the specific heat.  
466

#### 467 D) Electrical conductivity and viscosity

468 Fig. 8 (B) shows a parametric study on electrical conductivity and its effect on the temperature  
469 profile. The extreme limits of its value were used. Three values were chosen (0.05, 0.125, and 0.2  
470 S/m), It can be seen that the higher the electrical conductivity, the higher was the temperature gradient  
471 between  $T_1$  and  $T_2$ . **Electrical conductivity plays a major role in the efficiency of the heating process,  
472 especially for sweet products, which have a lower electrical conductivity than salty products. Further  
473 investigations should be carried out to assess the influence of the cake batter formulation on this  
474 property.** The same study was performed on viscosity: three values were studied (40, 25, and 13 Pa. s),  
475 namely the limits of variation of the batter viscosity between 20°C and 95°C. The results showed no  
476 significant impact of viscosity on the temperature profile.

477 In summary on the impact of batter properties, this study showed the importance of precision in the  
478 determinations of electrical conductivity, specific heat, density, and thermal conductivity, as they have  
479 a strong impact on the batter temperature profile. Among all the thermophysical properties of the cake  
480 batter, thermal conductivity appeared to be the one that most influenced the temperature gradient (the  
481 difference between  $T_1$  and  $T_2$ ). By increasing this property, the temperature gradient decreased, but this  
482 change was not sufficient to reverse the temperature profile (the temperature distribution inside the  
483 nozzle). Furthermore, it was difficult to control these properties, because they are temperature  
484 dependent, making the solution impossible.

485 To summarize this section, it seemed to be difficult to change the temperature profiles by adjusting  
486 the thermophysical properties of the cake batter. Hence, this situation led to the idea of studying the  
487 process parameters.

#### 488 E) Nozzle dimensions

489 Further work, based on parametric studies, was carried out to assess the effect of nozzle geometry  
490 on the temperature profile. To do this, different aspect ratios ( $\alpha = d_{elect}/l$ ) and nozzle dimensions were  
491 studied. The temperature distribution at the nozzle exit was determined for several electric field  
492 strengths, for the same flow velocity  $V_2 = 4$  mm/s. Three channel cross sections were used (presented  
493 in Table 4) to determine the effect of nozzle dimensions on the heterogeneity of the temperature along  
494 the nozzle outlet along the y-axis.  
495

496 **Table 4**  
497 Dimensions of nozzle sections.

	$l$ (cm)	$d_{elect}$ (cm)	$\alpha$
<b>Section 1</b>	2	0.5	0.25
<b>Section 2</b>	4	1	0.25

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Fig. 9 shows the evolution of the nozzle outlet temperature, at 65 V/cm (A) and 100 V/cm (B), for three different nozzle sections, as a function of the dimensionless number  $y^* = y/(l/2)$ . When the voltage was increased from 65 V/cm to 100 V/cm, for the same nozzle section,  $T_1$  and  $T_2$  increased by 22% and 23.5%, respectively. It can be observed that the increasing rate of temperature  $T_2$  was higher than that of  $T_1$ , which means that, at a stronger electric field, a higher temperature gradient was produced for the same section.

Fig. 9 (A) shows the effect of the nozzle section, at 65 V/cm, on the outlet nozzle temperature. Section 1 (green line), section 2 (red line), and section 3 (blue line) are presented, where section 1 and 2 had the same aspect ratio but different nozzle dimensions. At 65 V/cm, the temperature difference between  $T_1$  (center) and  $T_2$  (wall), for an aspect ratio of 0.5 (section 3), 0.25 (section 1), and 0.25 (section 2), were equal to 13 °C, 8.5 °C, and 15 °C, respectively. For 100 V/cm, the temperature differences between  $T_1$  (center) and  $T_2$  (wall) were equal to 18 °C, 12.5 °C, and 20 °C, respectively (Fig. 9B).

In summary, for different voltages, it was found that the aspect ratio affected temperature heterogeneity along the y-axis. For the same nozzle width  $l$  (section 1 and section 3), if the distance between the electrodes ( $d_{elect}$ ) was decreased from 1 to 0.5, then the temperature difference also decreased from 13 °C to 8.5 °C, at 65 V/cm. In addition, for the same distance between the electrodes (section 2 and 3), when the nozzle width  $l$  was increased from 2 to 4, the temperatures difference increased from 13°C to 15°C, for 65 V/cm. Finally, for all the voltages, it was found that section 1 was the most efficient, as it yielded the minimal temperature difference for different voltages, but did not provide a flattened or reversed temperature profile (warmer temperature at the center than in the corners of the channel), which was our final goal, in order to print and pre-bake at the same time.

According to the parametric study, we observed that it was difficult to obtain a homogeneous temperature along the y-axis, by adjusting the dimensions of the nozzle section. Further investigations should be carried out with different configurations because some modifications could result in a reduction of the overheating as in (Shynkaryk & Sastry, 2012). Cooling of the containment walls of the nozzle, which is the practice for some larger industrial ohmic heating systems, could also be part of the solution (Quarini, 1995). As OH is based on volumetric heating from the inside to the outside, a simulation was performed, involving the cooling of the nozzle walls along the y-axis. The simulation in stationary mode, for 4 mm/s and 65 V, showed that we had to remove almost 840 W/m<sup>2</sup>, which was equal to 4% of the total heat power produced in the product, on each side of the walls of the nozzle, in order to make the temperature almost homogeneous along the y-axis.

533 **4. Conclusion**

534 A new, innovative system was developed to study the possibility of pre-baking a cake batter in  
535 continuous flow, by implementing OH in a rectangular channel (nozzle). The study consisted of three  
536 main parts, namely the characterization of the thermophysical and electrical properties of the batter,  
537 experimental investigations, and CFD modeling of the experimental system, to better understand the  
538 links between the geometry of the printing nozzle, the physical properties of the batter, and the  
539 temperature distribution in the flow at the exit of the nozzle.

540 The experimental results showed considerable temperature heterogeneity along the y axis, which  
541 led to clogging at high temperatures. In-flow ohmic heating and batter baking resulted in heterogeneity  
542 in the temperature profile, and the temperature gradient in continuous ohmic heating was reversed,  
543 compared to the results obtained in batch conditions. Good correlation between results from  
544 simulations and experiments was observed, showing that the temperature heterogeneity along the y-  
545 axis was greater than that along the x-axis (electrodes distance). This finding was explained as being  
546 due to the presence of hot spots at the nozzle corners and electrode edges, where the electric field was  
547 higher than in the rest of the section of the nozzle. Also, the fact that the electrodes represented cold  
548 spots could explain why hot spots were found close to the corners of the nozzle, in the vicinity of the  
549 lateral walls, which were better insulated against heat transfer than were the electrode surfaces.

550 Different nozzle sections with aspects ratio of 0.25 and 0.5 (electrodes distances/width of nozzle)  
551 and voltage conditions were considered with the objective of mitigating the temperature gradient at the  
552 exit of the nozzle. The aspect ratio was found to affect temperature heterogeneity along the y-axis  
553 (channel width); decreasing the electrodes distance reduced the temperature gradient, while increasing  
554 the channel width yielded an increase of the temperature difference between the cold and the hot spots.  
555 Further investigations should be carried out with different nozzle-ohmic heating configurations, in  
556 order to find a suitable geometry capable of providing cake batter injection and pre-baking, without  
557 nozzle clogging. Short additional baking will be necessary to finish setting the structure after  
558 deposition. The effect of ohmic heating on food texture could be studied in the future once the  
559 partial baking process in a channel has been fully achieved. Once this will be achieved, application  
560 to 3D printing will be doable.

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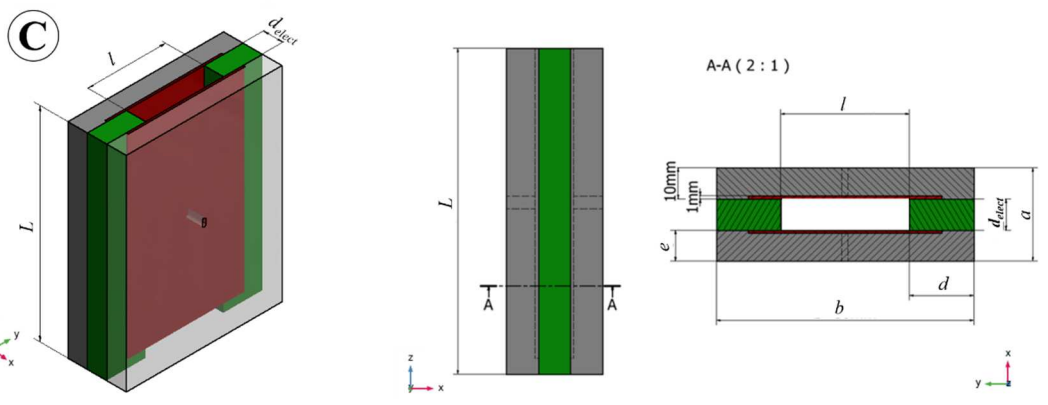
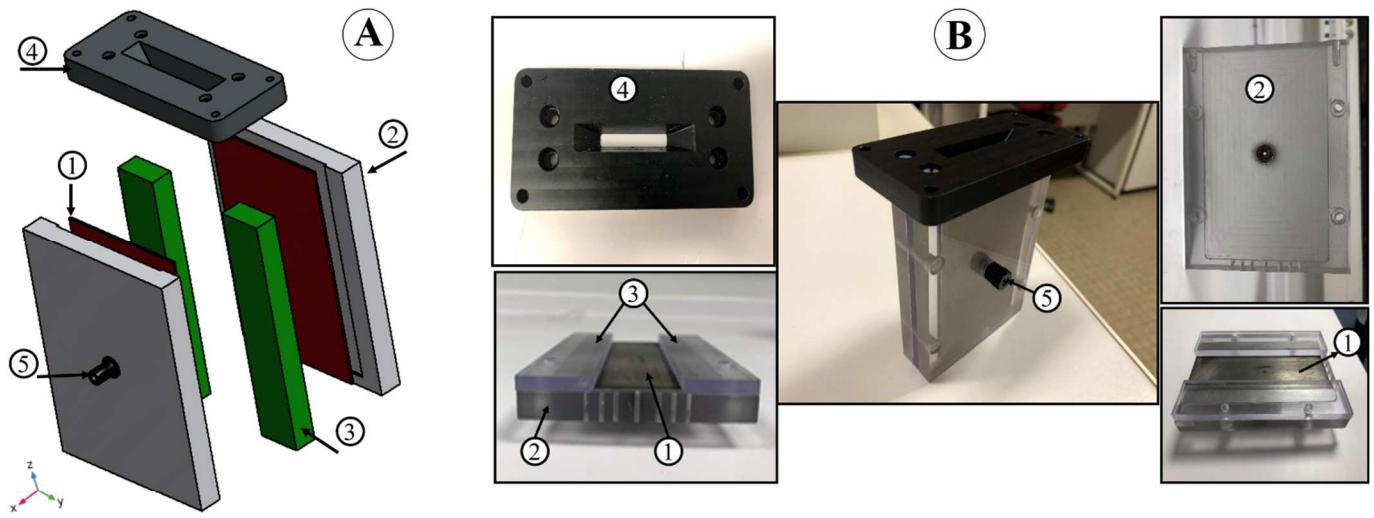
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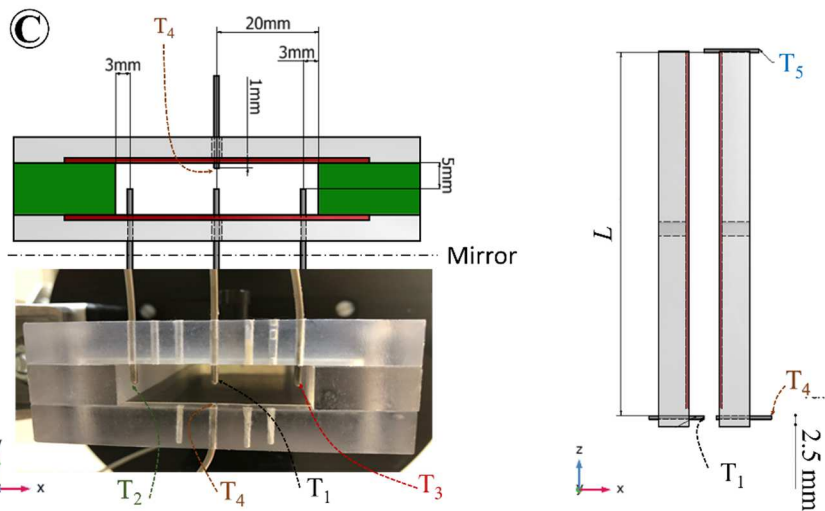
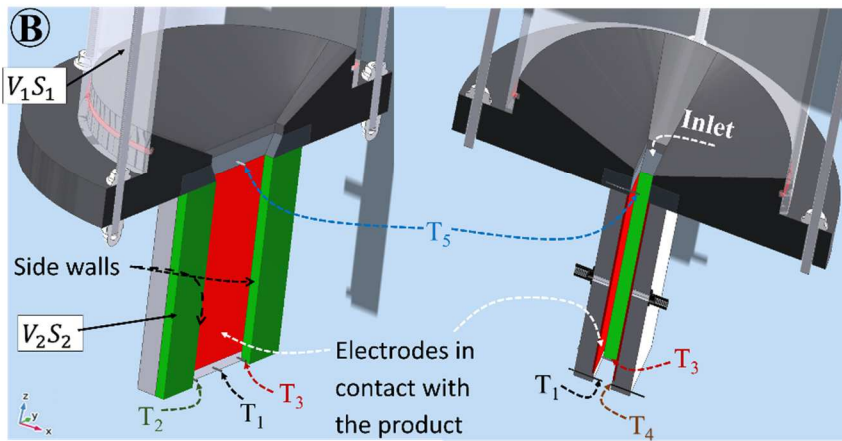
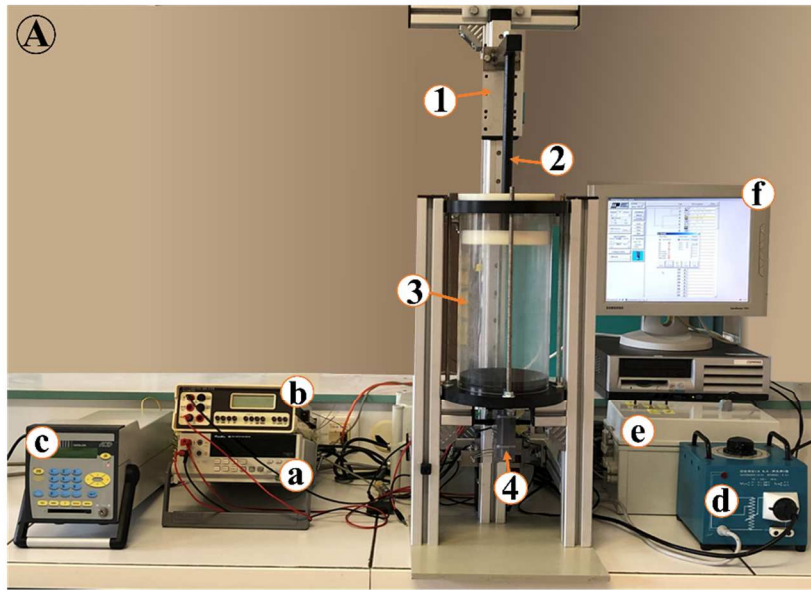
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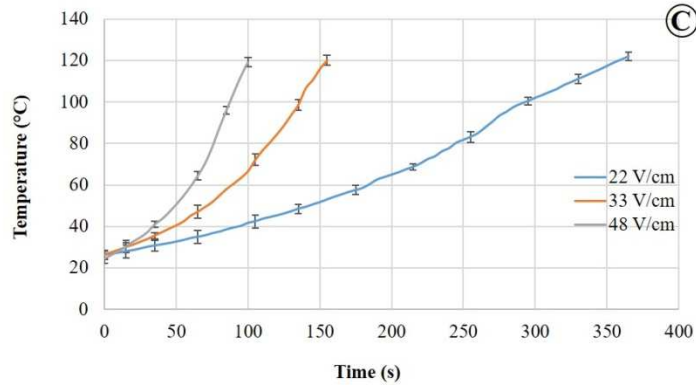
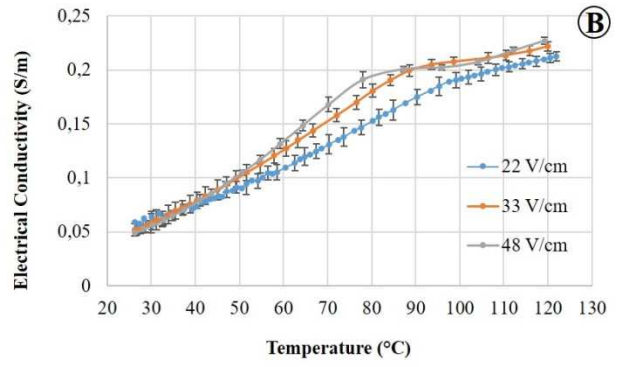
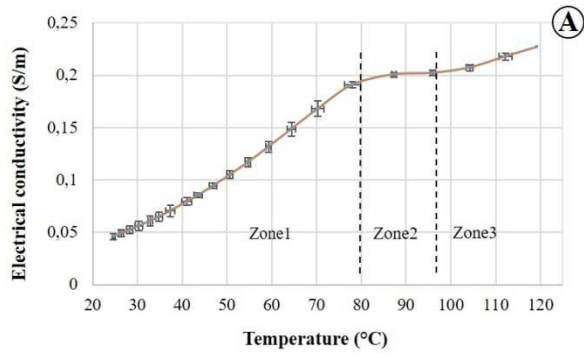
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**Fig. 1.** (A) Schematic of 3D nozzle geometry, consisted of ; (1) electrodes (red), (2) plates (grey), (3) beam (green), (4) channel support (black), (5) electric relays. (B) Different parts of the reel nozzle, same numbers as in (A). (C) Section views of the rectangular channel, distance between electrodes ( $d_{elect}$ ) along the x-axis, nozzle width ( $l$ ) along the y-axis; nozzle length ( $L$ ) along the z-axis.

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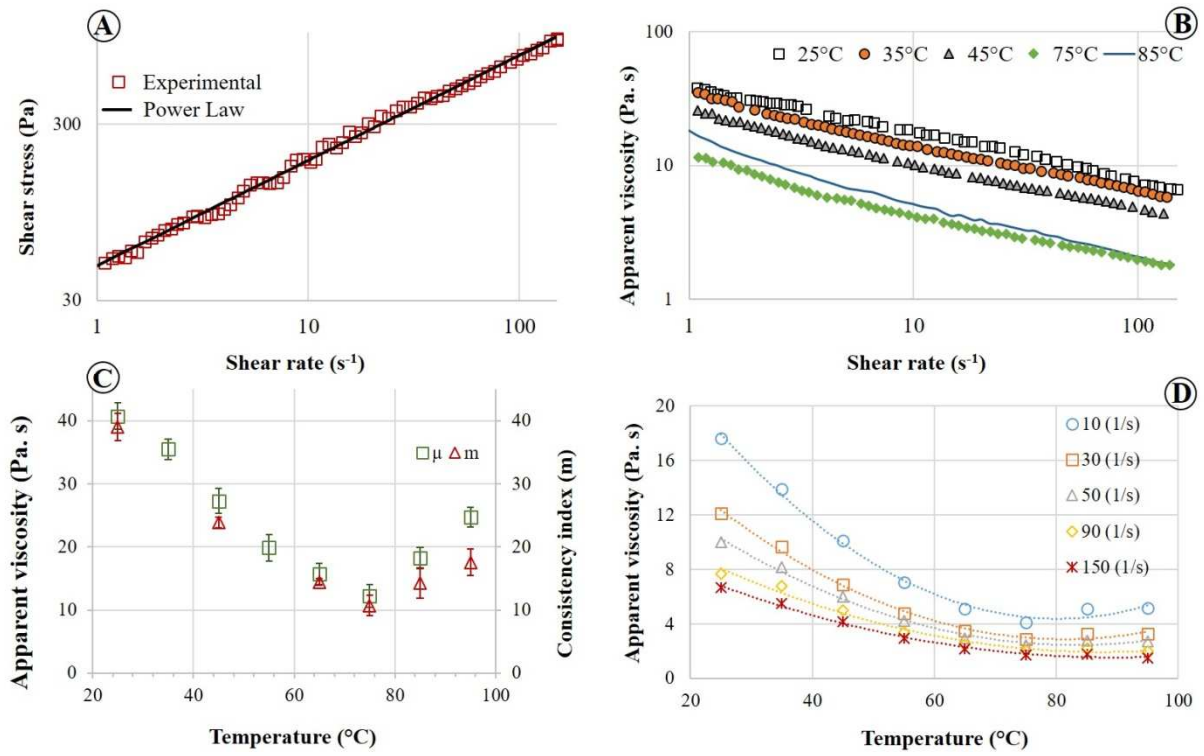
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 703 **Fig. 2.** (A) Overall installation of the system used for experiments, composed of mechanical parts: (1) drive motor,  
 704 (2) piston, (3) cylindrical container, and (4) nozzle with ohmic heating, and electrical parts: (a) voltmeter, (b)  
 705 ammeter, (c) data logger, (d) autotransformer, (e) motor controller, and (f) PC. (B) **3D views of the location of the**  
 706 **thermocouples in the product along the nozzle;**  $V_1$  and  $S_1$  are the velocity of the batter and surface of the cylindrical  
 707 container (3),  $V_2$  and  $S_2$  are the velocity of the batter and surface of the nozzle (4). (C) Location of each  
 708 thermocouple at the nozzle outlet and inlet.



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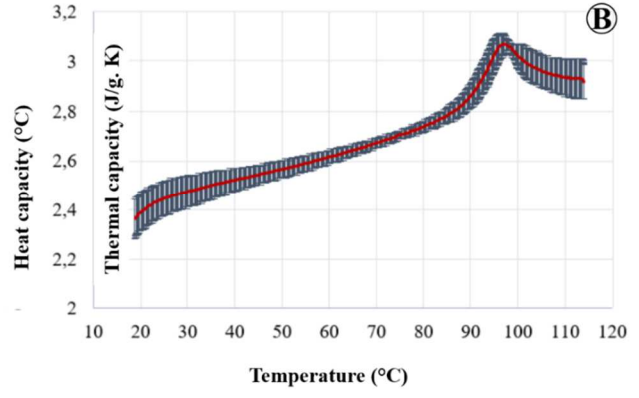
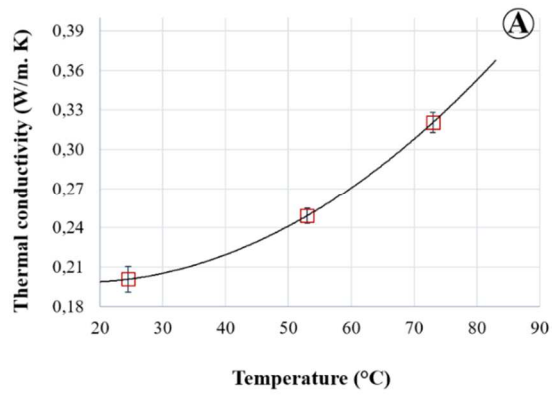
**Fig. 3.** (A) Electrical conductivity of the batter vs. temperature at a voltage gradient of 50 V/cm, (B) electrical conductivity of the batter vs. temperature at three different voltage gradients (22 V/cm (100 V), 33 V/cm (150 V), and 48 V/cm (200 V)) and (C) evolution of core temperature vs. time for the three different voltages.

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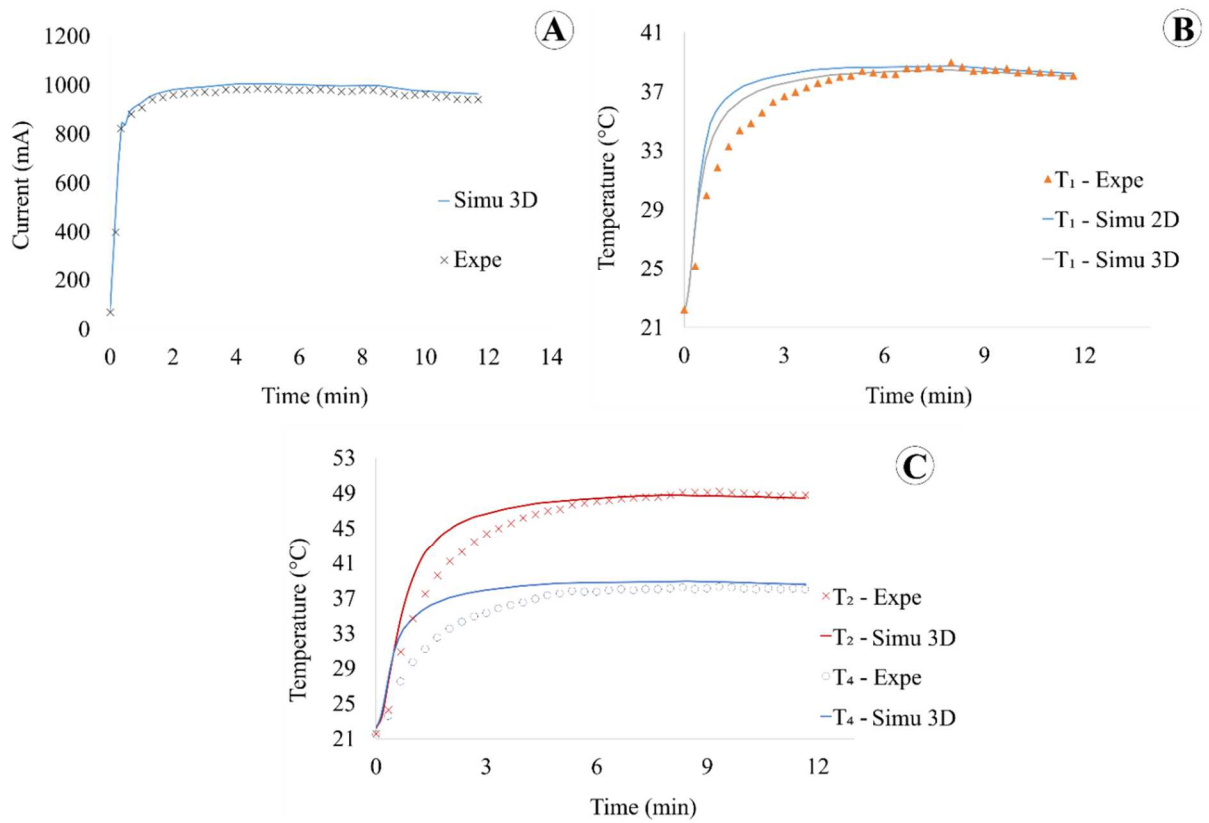
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**Fig. 4.** (A) Evolution of the batter shear stress vs. shear rate at 25 °C, in logarithmic form, (B) cake batter apparent viscosity vs. shear rate at five different temperatures, in logarithmic form, (C) consistency index and apparent viscosity at 1  $s^{-1}$  vs. temperature, and (D) apparent viscosity vs. temperature at five different shear rates.



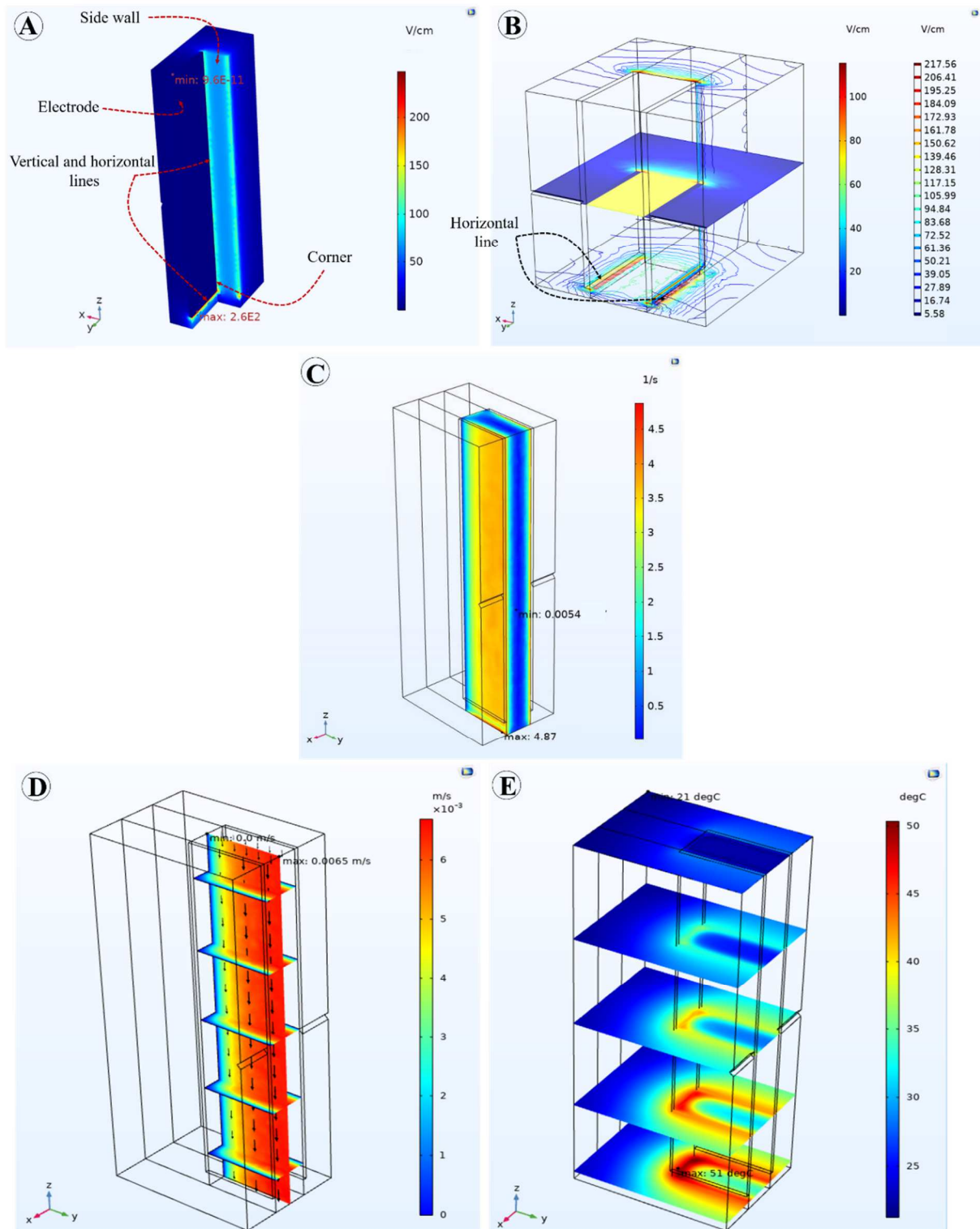
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**Fig. 5** (A) Thermal conductivity and (B) heat capacity of the cake batter vs. temperature.



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 734 **Fig. 6** Validation of the model: experimental and modelled data for (A) the current evolution between electrodes,  
 735 (B) temperature measurements at the outlet ( $T_1$ ), and (C) temperatures  $T_2$  and  $T_4$  vs. time at the nozzle outlet for  
 736 12 min of injection at 65 V.

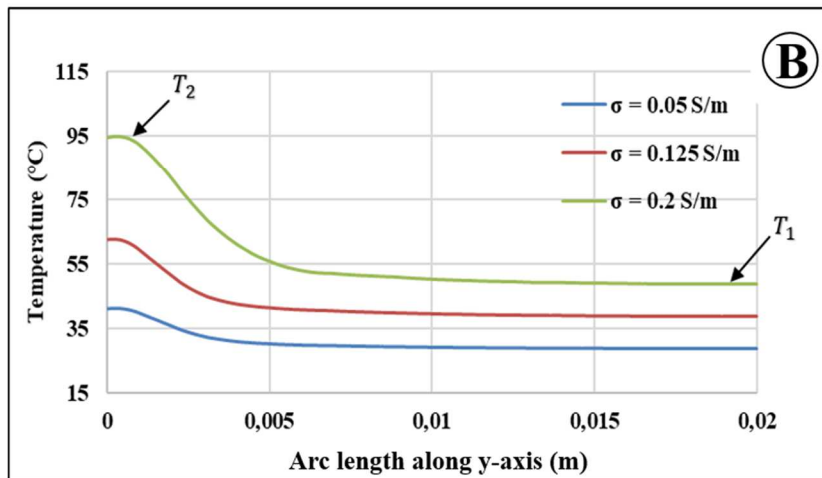
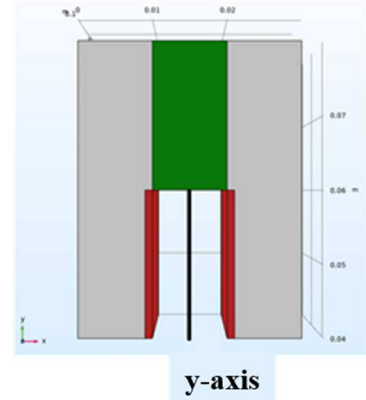
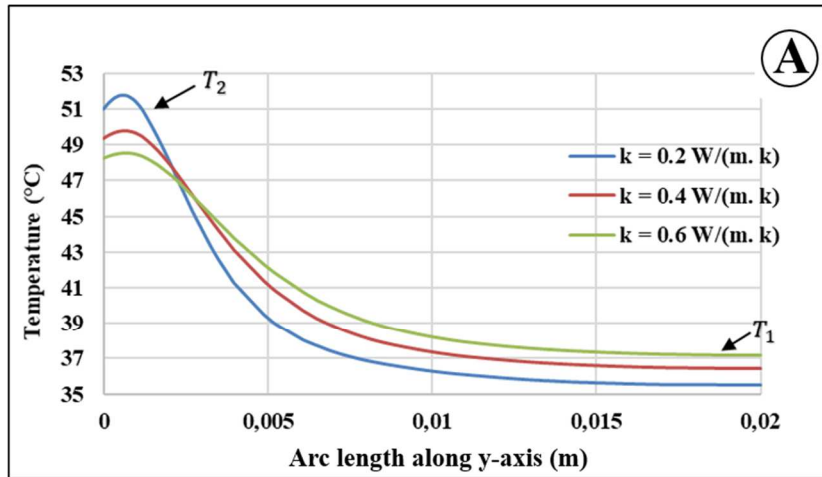
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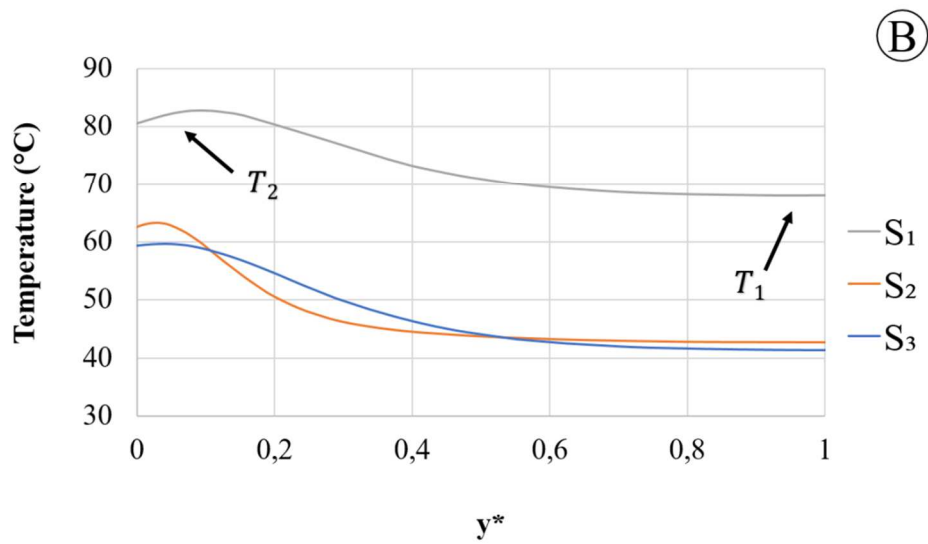
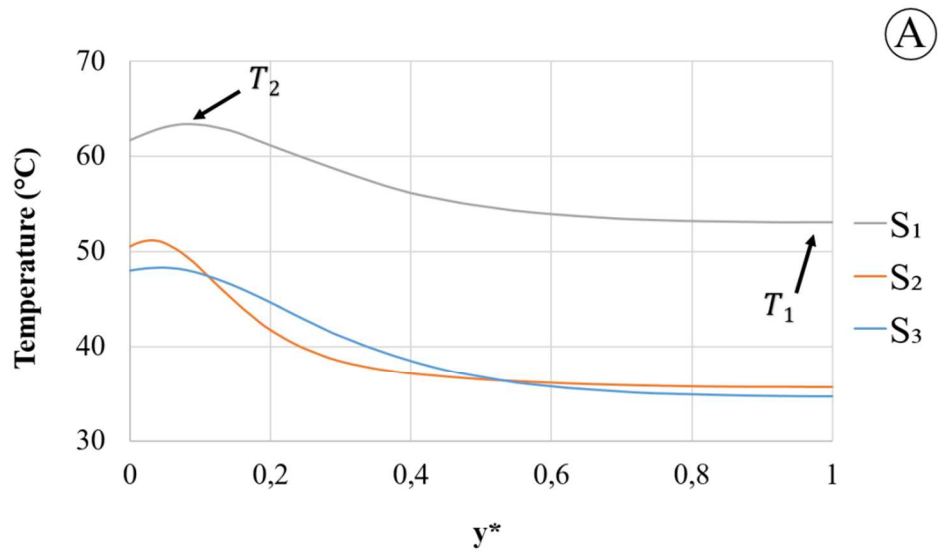
**Fig. 7.** Plots for the cake batter after 12 min heating at 65 V and 4 mm/s, for (A) volume electric field; (B) electric field in the xy plane; (C) volume shear rate; (D) velocity along the y axis; and (E) temperature distribution in the nozzle in the xy plane at different location along z axis.





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**Fig. 8.** Parametric study of (A) thermal conductivity for 65 V – 4 mm/s, (B) electrical conductivity for 65 V- 12 mm/s, and their effects on the outlet temperature profile along the y-axis.



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**Fig. 9.** Temperature profiles vs non-dimensional coordinate  $y^* = y/(l/2)$  for three different nozzle sections:  $S_1 = 1 \text{ cm}^2$ ,  $S_2 = 4 \text{ cm}^2$  and  $S_3 = 2 \text{ cm}^2$  with aspect ratios ( $\alpha = d_{elec}/l$ ) equal to 0.25, 0.25 and 0.5 respectively, and for two voltages: (A) 65 V/cm, and (B) 100 V/cm, at a velocity of 4 mm/s.