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# A versatile simulator for defrosting, convective and boiling drying during hot air frying process

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## Abstract

A versatile simulator takes into account four major stages: defrosting, warm-up, and convective and boiling drying. This dynamic model considers that a frozen prefried french fries has three compartments: a central compartment (#1) with high water content, a peripheral compartment (#3) corresponding to the prefried dry crust, and an intermediate compartment (#2) appears during frying, filled with water vapour. Convective and boiling drying and freezing are modeled. The predicting model has been identified based on the triplicates of three different modalities. The resulting predictions have been experimentally validated.

**Keywords:** Frying; Modelling; Heat transfer; French fries; Mass transfer

## 1. Introduction

Gouyo et al. (2021b) published results of X-ray microtomography showing that prefried frozen French fries have a moist and soft core, and a crispy outer dry crust of approximately 0.5 — 1.5 mm (Bouchon and Aguilera, 2001; Pedreschi and Aguilera, 2002). French fries are popular potato products because of their structure and attractive texture (Garayo and Moreira, 2002). Deep-fat frying can be defined as a process of drying and cooking through contact with hot oil. This process essentially consists of soaking the product in hot vegetable oil at a temperature above the boiling point of water, typically 150—180 °C. These frying conditions lead to high rates of heat and mass transfer, causing water loss and oil uptake, with consequent changes in taste, texture and colour properties.

Two disadvantages of consuming deep-fat fries, and in general all deep-fat fried products, is related to their high fat content (about 20 — 40 g oil/100 g fat-free dry

matter) (Garayo and Moreira, 2002) and to the time it takes to fry them.

Hot-air fryers, recently developed, prevent the high oil uptake during fries. The objective is to reduce oil uptake when consuming french fries. The hot-air frying aims to produce a "fried product" by sparging, essentially, hot-air around the material instead of immersing it in hot oil (Andrés et al., 2013; Heredia et al., 2014; Sansano et al., 2015; Teruel et al., 2015; Tian et al., 2017). Their hot air bring surface of french fries to the boiling point, creating a fine mist of oil droplets in hot-air and the product. They may combine forced convection, radiation, and/or conduction.

The structure of the core and of the crust of a frying product is affected by the temperature kinetics and the process. Therefore, a better understanding of the heat and mass transport is a useful step to improve hot-air fries. This understanding will be based on a mathematical model, based on fundamental physical principles.

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The aim of this work is a dynamic model for the hot-air frying of frozen pre-fried french fries, simulating both heat and mass transfer.

## 2. State of the art of frying models

Various models have been developed to describe deep-fat frying in the past. However, the changes occurring during frying are difficult to model, as several interdependent factors must be taken into account. It is important to identify the structural changes during the different stages of the process in order to understand the quality changes that occur during frying. The mechanism of water loss during frying is complex and transport by molecular diffusion, capillary and pressure are often considered. The complexity ranges from simple empirical equations to complex numerical models. All models have their own advantages in describing the water loss during frying.

A first-order kinetic model in which water loss was proportional to moisture content was considered by Gupta et al. (2000) and Krokida et al. (2001). These models consider the frying material (usually very thin products such as chips), as a single zone in which the presence of crust is neglected. The single zone model explains the heat and mass transfer equations for the entire product without considering the difference between the core and the crust.

Models with two concentric compartments were also used (Farkas et al., 1996; Farid and Chen, 1998; Bouchon and Pyle, 2005). The crust and core are treated as two regions separated by a moving boundary (front model) and a pressurized flow in the crust region is included. Each region evolves during frying, the crust thickens and the core thickness decreases. In each region, simultaneous heat and mass transfer occurs, resulting in thermal and water content gradients.

We also find in the literature multiphase porous media models (Xiong et al., 1992; Ni and Datta, 1999; Yamsaengsung and Moreira, 2002) and compartmental dynamic models (Courtois et al., 1998; Costa and Oliveira, 1999) to describe deepfat frying. All these models take into account the latent heat of vaporization and the heat transport, but the transfer of matter by entrainment is often neglected. From the point of view of the boundary conditions a constant  $h_c$  coefficient is often used. Similar approaches to frying models are generally used in the process of entrainment drying. In the case of entrainment drying of a biological product, it is usually the internal material transfer that is the limiting phenomenon. In the case of boiling drying, it is rather the heat transfer rate that limits the drying rate (Bonazzi and Bimbenet, 2003).

During baking in hot air ( $T_a > 100^\circ\text{C}$ ) in an oven of products such as bread or cake dough for example, two heating periods were generally considered in the models with simultaneous heat and water transfer: (i)

period of product heating to  $100^\circ\text{C}$  and (ii) period of crust and core formation (Vries et al., 1988; Zanoni et al., 1994; Sablani et al., 1998; Lostie et al., 2002; Flick et al., 2015). According to Zanoni et al. (1994), during the period (i), water is vaporized at the surface of the product and the drying rate is controlled by diffusion of liquid water from the core to the surface. During the period (ii), a water vaporization front appears at  $100^\circ\text{C}$  and moves inwards resulting in the formation of two distinct regions: the crust and the core. The drying rate during this period is controlled by heat conduction through the surface dry layer.

A strong similarity with hot air frying appears in the description of the bread baking process, despite the presence of other complex phenomena such as fermentation and the presence of gases in the bread doughs. The approaches to modeling the period (ii) during bread baking are very similar to those for deep frying.

We can cite two scientific publications about hot-air frying: (Andrés et al., 2013) compares kinetics of mass transfer and volume changes in hot-air frying and deep-fat frying at  $180^\circ\text{C}$ ; Our work (Gouyo et al., 2021a) is the only one about hot-air frying with a dynamic model, and we present here its main results.

In this study, by microstructure analysis of french fries, Gouyo et al. (2021b), the french fries were modelled as three concentric compartments in series, with heat and mass resistances at interfaces only.

## 3. Materials and methods

### 3.1. Raw materials and frying equipment

The experiments were carried out with commercial frozen pre-fried (Vauvre et al., 2014; Aguilera and Gloria-Hernandez, 2000) french fries (Mc-Cain Tradition) stored in a cold room at  $-18^\circ\text{C}$ . These French fries (McCain tradition) are designed for domestic deep fat frying of 5 to 10 minutes. A primary selection of frozen french fries was done, according to (1) being straight, (2) with a square section of  $9 \times 9$  mm exactly, and (3) long enough to be resized to 60 mm in length with a specific cutter.

Hot-air fryer equipment (Airfryer Philips XL HD9240/90, Avance Collection, Amsterdam, The Netherlands) with a power of 2100 W was used. A specific fryer's control system allowed to adjust the air velocity and to add an extra radiative heating component (halogen, Suney 012072, 1000 W). The air velocity was measured above the basket with an anemometer (MiniAir 64 Mini, OmniInstruments, UK). This modification allowed for three different operating conditions in terms of the intensity of energy input and the mode of input. The fryer was instrumented with a power meter (PM231 – Powermeter, brennenstuhl, CHINA) connected to the energy sources (electrical resistance and radiative heating source). The fryer was placed on a scale (Sartorius CPA34001S, France) that allowed

recording the water loss.

In the single bed of 0.200 kg of french fries, the fries hardly touch each other. Triplicates were performed for each experiment.

### 3.2. Temperature measurements

Two thermocouples were inserted in the potato, either at the surface ( $T_s$ ) or at the core ( $T_{co}$ ). Infrared thermometers (MLX90614KSF-ACC, Melexis, Corbeil-Essonnes, France) were also used to measure the surface temperature ( $T_{IR}$ ). The air temperature ( $T_a$ ) away from the french fries was also measured with a type J thermocouple placed in the Airfryer.

## 4. Model development

### 4.1. Justification for three compartments

The experimental observation of frozen french fries during hot-air frying by X-ray microtomography by Gouyo et al. (2021b) shows a central compartment with a pre-crust formed at the periphery of the frozen french fry. At the end of frying, three distinct compartments were observed: the core has shrunk while an intermediate gaseous one has appeared.

These findings split the french fries into 3 different compartments (figure 1):

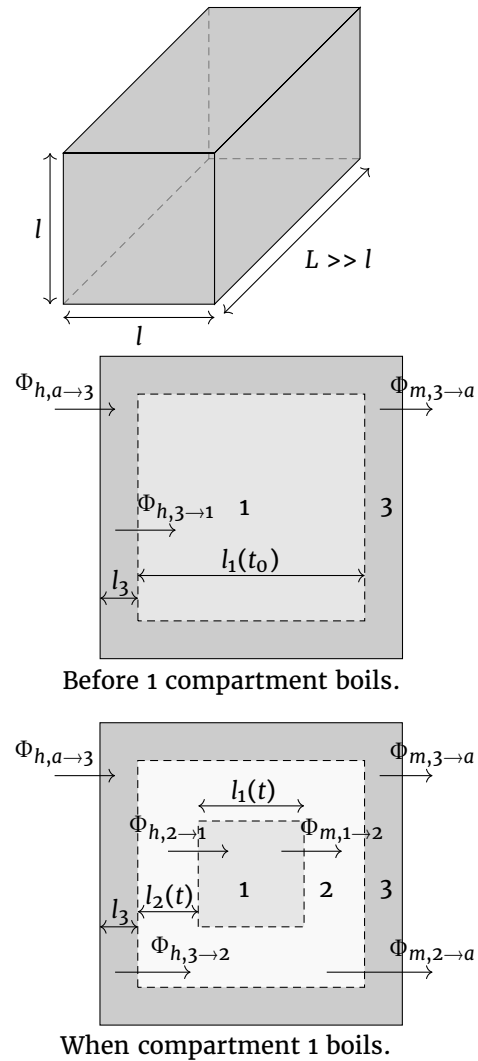
1. the central compartment #1, containing wet fat-fattened matter, whose ice content decreases from 15% to 0%, then whose temperature  $T_1$  reaches boiling point including ebullioscopic elevation, then its water content decrease starting from 75%, creating a volume shrinkage.
2. the medium compartment #2 is filled of  $m_2$  (in kg; initially null) of vapor possibly superheated at temperature  $T_2$  that escapes from compartment #1 through compartment #3.
3. the peripheral compartment #3 corresponds to the crust compartment formed while pre-frying the french fries. It is a porous wet fat-fattened matter, with very low water content and higher oil content.

### 4.2. General assumptions

To minimise computational effort, Gouyo et al. (2021a)'s assumptions were kept. The evolution of the different successive mechanisms taken into account in the model are listed as well.

### 4.3. Shrinkage of compartment #1

By microstructure analysis of french fries, Gouyo et al. (2021b) showed that the porosity created in the frozen french fry matched the water loss of the product. The



**Figure 1.** French fries are seen as infinite rectangular parallelepiped with 3 concentric shells or compartments (the second being negligible before compartment 1 boils).

**Table 1.** Values and variations of state variables for compartments #1, #2 and #3.  $\nearrow$ : grow,  $\searrow$ : decrease. Unless written otherwise, states keep their previous values. "drying" refers to convective drying. "boiling" refers to drying by boiling.

Step	#1	#2	#3
warm-up #1 and #3	$T \nearrow; P=P_a$	$V=0$	$T \nearrow$
warm-up #1, defrost #3	$T \nearrow$		$T=0; I \searrow; W \nearrow$
warm-up #1, drying #3	$T \nearrow$		$T \nearrow; I=0; W \searrow$
defrost #1, drying #3	$T=0; I \searrow; W \nearrow$		$T \nearrow; W \searrow$
warm-up #1, drying #3	$T \nearrow; I=0$		$T \nearrow; W \searrow$
warm-up #1, boiling #3	$T \nearrow$		$T \nearrow; W \searrow$
boiling #1 and #3	$T \nearrow; V, W \searrow; P=P_2$	$T, V, P, m \nearrow$	$T \nearrow; W \searrow$

same analogy was made for the water loss in compartment #1, as done by Gouyo et al. (2021a).

#### 4.4. Convective drying (boundary equations)

Before boiling happens, water flux density (in kg water per second and square meters) from compartment #3 to hot-air (shown in figure 1) is expressed as

$$\phi_{m,3 \rightarrow a} = k_{3,a} \cdot \frac{M_w}{R} \cdot \left( \frac{Aw_3 \cdot P_{vsat}(T_3)}{T_3 + 273.16} - \frac{RH_a \cdot P_{vsat}(T_a)}{T_a + 273.16} \right) \quad (1)$$

where  $M_w$  is the molar mass of water,  $R$  is the ideal gas constant,  $k_{3,a}$  the mass transfer coefficient ( $m \cdot s^{-1}$ ), and where the water activity  $Aw$  satisfies

$$X = \frac{14.11 - 5.013 \cdot 10^{-2} \cdot T}{100} \cdot \left( \frac{Aw}{1 - Aw} \right)^{0.124 + 2.063 \cdot 10^{-3} \cdot T} \quad (2)$$

#### 4.5. Boiling

Water activity, product temperature and water content are linked by above equation (1). When boiling, they are also linked by this equation relying on Clausius-Clapeyron relationship (Murray, 1967):

$$Aw \cdot P_{vsat}(T) = P \quad (3)$$

where  $P_{vsat}(T) = 10^{2.7858 + \frac{7.5 \cdot T}{T + 237.3}}$  Pa.

At known pressure, water content and product temperature are linked by above two equations in a boiling compartment, hence their derivative are also linked (Gouyo et al., 2021a).

When compartment #1 boils, compartment #2 emits water vapor to air which is "filtrated" by compartment #3 following Darcy's law (Loncin and Merson, 1979). The equation is:

$$\phi_{m,2 \rightarrow a} = \rho_v \cdot \frac{\kappa_v}{\mu} \cdot \vec{\nabla} P = \rho_v \cdot \frac{\kappa_v}{\mu \cdot l_3} \cdot (P_2 - P_a) \quad (4)$$

#### 4.6. Heat transfer

Assuming convective drying, heat flux densities can be written simply as:

$$\phi_{h,a \rightarrow 3} = h_{3,a} \cdot (T_a - T_3) \quad (5)$$

Prior to compartment #1 boiling, there is no compartment #2 and direct heat transfer between compartment #1 and #3 is done by conduction:

$$\phi_{h,3 \rightarrow 1} = h_{1,3} \cdot (T_3 - T_1) \quad (6)$$

When compartment #1 is boiling then, heat transfer is indirect, by convection through compartment #2:

$$\phi_{h,2 \rightarrow 1} = h_{1,2} \cdot (T_2 - T_1) \quad (7)$$

$$\phi_{h,3 \rightarrow 2} = h_{2,3} \cdot (T_3 - T_2) \quad (8)$$

Note: It was assumed that  $h_{1,2} = h_{2,3}$ .

These equations (1)-(8) are taken from publication Gouyo et al. (2021a). This publication also details thoughtfully their effect on heat and mass balances.

#### 4.7. Frozen condition

Assuming no sublimation, the ice first becomes liquid water then vapour. To keep the model the simplest possible, we assumed that defrosting occurred at 0°C while, in practice, food defrosting takes place at temperatures below 0°C that unfortunately depend on the solid water content. Moreover, it was assumed that no (negligible) drying occurs in a compartment #i where ice content  $I_i > 0$ .  $X_i$  is the total water content (ice+liquid water) with respect to the dry fatted matter (dry matter + oil).  $I_i$  is the frozen water content, hence the liquid water content is equal to  $W_i = X_i - I_i$ .

## 5. Results and discussion

On Matlab (version 2017b, Mathworks, USA), simulation time was one second using  $10^{10}$  bytes of memory (RAM) on a 64 bit computer with 2.30 GHz Intel(R) Core (TM) i5-62004 CPU®.

The identified heat and mass transfer coefficients are shown in Table 2. Available literature is scarce about mass transfer coefficient for potato products. In comparison, the identified values for  $k_{3,a}$  are reasonable ( $3.58 \cdot 10^{-3}$  to  $4.10 \cdot 10^{-3} m \cdot s^{-1}$ ), but remain high compared to the values found in the case of conventional drying. Comparing the value of the mass transfer coefficient identified in this study to literature is difficult due to different conditions of the drying process. For instance, most authors (Miketinac et al., 1992; Białobrzewski, 2007; Dhalsamant et al., 2017) use a difference in water content in mass as the driving force for the water flux density, and the values of the mass transfer coefficient obtained are generally in the magnitude of  $10^{-5} - 10^{-4} m \cdot s^{-1}$ .

Table 2 shows that despite the different modes of heat transfer, the global heat transfer coefficients remains still quite similar. High Convection frying mode provides a higher global heat transfer coefficient (High Convection,  $h_{3,a} = 96 W \cdot m^{-2} \cdot K^{-1}$ ). The higher is  $h_{3,a}$ , the faster  $T_3$  reaches 100°C and the higher it rises at the end of the frying process (figure 2). The heat transfer coefficients obtained are extremely far from the values

**Table 2.** Identified values for transfer parameters of heat ( $h_{1,3}$ ,  $h_{2,3}$ ,  $h_{3,a}$ ) and water ( $k_{3,a}$ ).

Parameter	Unit	Setting	Identified value
$h_{1,3}$	$\text{W}\cdot\text{m}^{-2}\cdot\text{K}^{-1}$	All	$254 \pm 9$
$h_{2,3}$	$\text{W}\cdot\text{m}^{-2}\cdot\text{K}^{-1}$	All	$202 \pm 5$
$h_{3,a}$	$\text{W}\cdot\text{m}^{-2}\cdot\text{K}^{-1}$	C-Standard	$66 \pm 1$
$h_{3,a}$	$\text{W}\cdot\text{m}^{-2}\cdot\text{K}^{-1}$	C-Standard-Halo	$73 \pm 1$
$h_{3,a}$	$\text{W}\cdot\text{m}^{-2}\cdot\text{K}^{-1}$	High Convection	$76 \pm 1$
$k_{3,a}$	$\text{m}\cdot\text{s}^{-1}$	C-Standard	$3.70 \cdot 10^{-3} \pm 0.06 \cdot 10^{-3}$
$k_{3,a}$	$\text{m}\cdot\text{s}^{-1}$	C-Standard-Halo	$4.10 \cdot 10^{-3} \pm 0.06 \cdot 10^{-3}$
$k_{3,a}$	$\text{m}\cdot\text{s}^{-1}$	High Convection	$3.58 \cdot 10^{-3} \pm 0.20 \cdot 10^{-3}$

that can be obtained in the deep-fat frying process, which can be up to 250 — 1000 ( $\text{W}\cdot\text{m}^{-2}\cdot\text{K}^{-1}$ ).

The internal transfer coefficients ( $h_{1,3}$  and  $h_{2,3}$ ) are four times the external transfer coefficients ( $h_{3,a}$ ), proving non limiting internal heat transfer.

A wide range of values for intrinsic permeability  $\kappa_V$  ( $10^{-10}$  to  $10^{-14}$ ) were tested, showing negligible effects on the model predictions.

Figure 2 displays the experimental and simulated evolution of french fry temperature during frying for the different geometrical points. The mean coefficient of variation for experimental temperature acquisition inside french fry was quite good ( $< 10^\circ\text{C}$ ) and root-mean-square error (RMSE) between experimental and predicted values was reasonable as well. As shown on 2, the air temperature close to the french fries rises rapidly to  $150^\circ\text{C}$ , then gradually reaches  $180^\circ\text{C}$  in less than 120 seconds of cooking. The evolution of the estimated- actual surface temperature is represented by an area between the  $T_{IR}$  measured by infrared and the  $T_s$  measured by a thermocouple. The surface temperature of the french fries increases relatively rapidly due to the high initial temperature difference between the hot-air and the frozen french fries. To reach  $100^\circ\text{C}$  on the surface of the french fry, it takes 100 to 120 seconds of frying as compared to 20 seconds for deep-fat frying (Achir et al., 2008; van Koerten et al., 2017). This is related to the higher heat transfer coefficient between oil and product.

The experimental core temperature shows a plateau around  $-4^\circ\text{C}$  at the beginning of frying, related to the core defrosting which occurs in less than 60 s. This plateau does not fit well with the simulation data because we assumed, to keep the model the simplest possible, that defrosting occurred at  $0^\circ\text{C}$ . This leads to a slight mismatch in the evolution of the simulated core temperature with the experimental core temperature. This assumption leads to a mismatch on temperature prediction that becomes negligible after the initial defrosting. The temperature increase in the core compartment (compartment #1) is delayed compared to the compartment #3, due to the resistance to heat transfer in the crust compartment (compartment #3). However, it reaches a value around  $100^\circ\text{C}$ , which is

maintained due to the high latent heat of evaporation of water which is present in large amounts in the core. This boiling temperature is reached in the core after about 120 s of frying.

The same plateau stagnation is observed for the surface temperature, but the duration is much shorter than for the core: the temperature increase is halved for a few seconds. The reason for this is the rapid dehydration of the external compartment #3. When this compartment is almost dried, it can no longer serve as an energy sink and its temperature  $T_3$  begins to rise.  $T_3$  is well below the temperature of the hot-air near to the french fries. This is because the water evaporating from deeper inside the fry still acts as a cooling source. This plateau can also be observed in the case of deep-fat frying (5 s) but the duration of the plateau is very short compared to hot-air frying (Farid and Kizilel, 2009; Lioumbas and Karapantsios, 2012; Lalam et al., 2013; van Koerten et al., 2017).

The model predictions of the temperature for the nominal frying condition (other conditions shown by Gouyo et al. (2021a)) are also shown in the figure 2, along the measurements. Actually, the temperature evolution in both the core (compartment #1) and the surface (compartment #3) are reasonably predicted for all data sets.

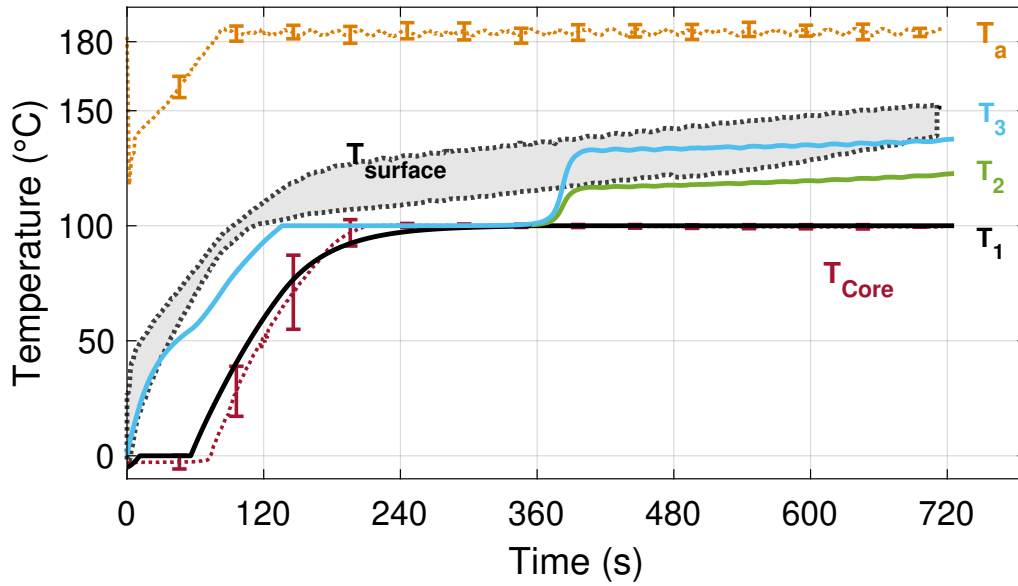
The model predicts a plateau at  $100^\circ\text{C}$  for temperature in the compartment #3 (Surface) for a few minutes. Non-invasive techniques (Touffet et al., 2020) showed that mechanical fractures and cracks (with cavitation) occur beneath the rigid (glassy) crust, justifying this discrepancy. The model assumes a crust thickness (compartment #3) is 1 mm. The state variable  $T_3$  is therefore an average of the temperature in compartment #3.

Analysis of predicted pressure evolution at the core of the french fry shows that  $P_2$  is very close to  $P_a$  (overpressure  $< 0.5$  kPa). This result supports the assumption that  $P_1 = P_a$ . During deep-fat frying, the overpressure  $\Delta P_a$  is larger than 30 kPa (Patsioura et al., 2016; Vauvre et al., 2014), which is way higher than what we observed in hot-air frying.

The model fitted very well the experimental mean water content. The assumption of convective drying of the compartment #3 before boiling is relevant, allowing to fit the first drying phase.

The global shrinkage of french fries is very small (Gouyo et al., 2021b): less than 10 % in both radial and longitudinal directions. Therefore the mechanical effect of heat and mass transfer was neglected. The only significant shrinkage is local and related to compartment #1, which considers that a volume of water lost in compartment #1 produces a shrinkage of the same volume (Gouyo et al., 2021b).

On the left of Figure 3, there is a stage (phase 1, the initial heating period) that corresponds to the rise the heating and the defrosting phase of the french

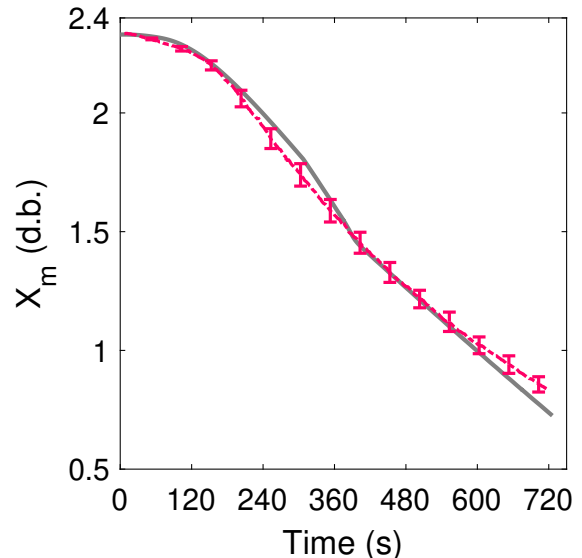


**Figure 2.** The measured evolution of temperatures in time plotted for the core ( $T_{core}$ ) and the surface of the french fries ( $T_{surface}$ ) in reference air-frying conditions.  $T_a$  = Air temperature in the area away from the french fries and  $T_1$ ,  $T_2$ , and  $T_3$  correspond to the temperatures of compartment #1, #2 and #3 respectively. Error bars represent twice ( $2\sigma$ ) standard deviation between replicates for data (dashed lines). The solid lines represent the model predictions.

fries. During this stage,  $T_3$  is raised to the boiling temperature of the water, after which evaporation begins. Obviously, this stage lasts more if the heat transfer coefficient ( $h_{3,a}$ ) is lower (observed only for the C-Standard condition). Dehydration of the product then follows, to reach a maximum around 300 to 400 s. The first dehydration slope observed on the drying curve (figure 3) corresponds to the evaporation phase in compartment #3 and the rise in  $T_1$ . After the water of compartment #3 has been evaporated, the temperature in compartment #3 will exceed the boiling point of water (figure 2). The second slope corresponds to the maximum water evaporation, coinciding with the boiling phase in the compartment #1. The rate of evaporation is at its maximum at this moment because both compartments #1 and #3 are boiling and there is more water in compartment #1. The highest value of the measured vapour flow density for the different hot-air frying conditions was relatively lower ( $2.5 \cdot 10^{-3} \text{ kg}\cdot\text{m}^{-2}\cdot\text{s}^{-1}$ ) than what was found in deep-fat frying ( $5 - 10 \cdot 10^{-3} \text{ kg}\cdot\text{m}^{-2}\cdot\text{s}^{-1}$ ) (Costa and Oliveira, 1999; Vitrac et al., 2002; Ziaifar, 2008; van Koerten et al., 2017). Despite the different evolution of the surface temperature for the three frying conditions, the evolution of the water loss remains very close.

## 6. Conclusion

A dynamic three-compartment model including heat and vapour transfer was developed. Taking into account four major stages: defrosting, warm-up, con-



**Figure 3.** Water loss as function of frying time for reference frying conditions. Experimental data (.....), predictions (—). Error bars represent twice ( $2\sigma$ ) standard deviation between triplicates.

vective drying and boiling drying. A good fit was observed with the experimental data of water content and core temperature evolution during hot-air frying. The model is hence accurate to simulate hot-air frying of frozen -pre-fried- french fries at different conditions and is considered relevant for the frozen pre-fried french fries from defrosting, warming-up to convective drying and boiling drying of the crust and boiling drying of the core. In addition, it allows to study the effect of variable air characteristics and different ways of providing energy to the product. The convective heat transfer coefficient between the external compartment and the hot-air is the most sensitive. We consider it can be a promising tool for the control and optimization of hot-air frying.

## 7. Acknowledgements

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